

# VARIABLE VOLUME SYSTEMS

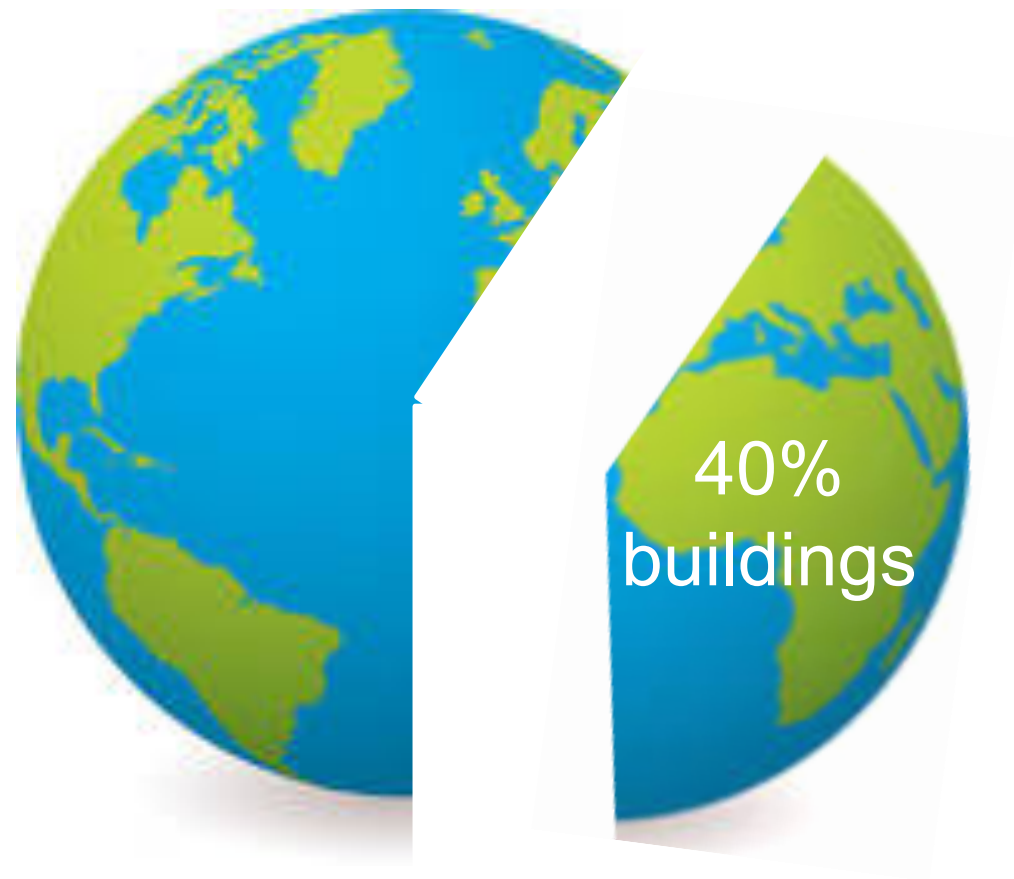
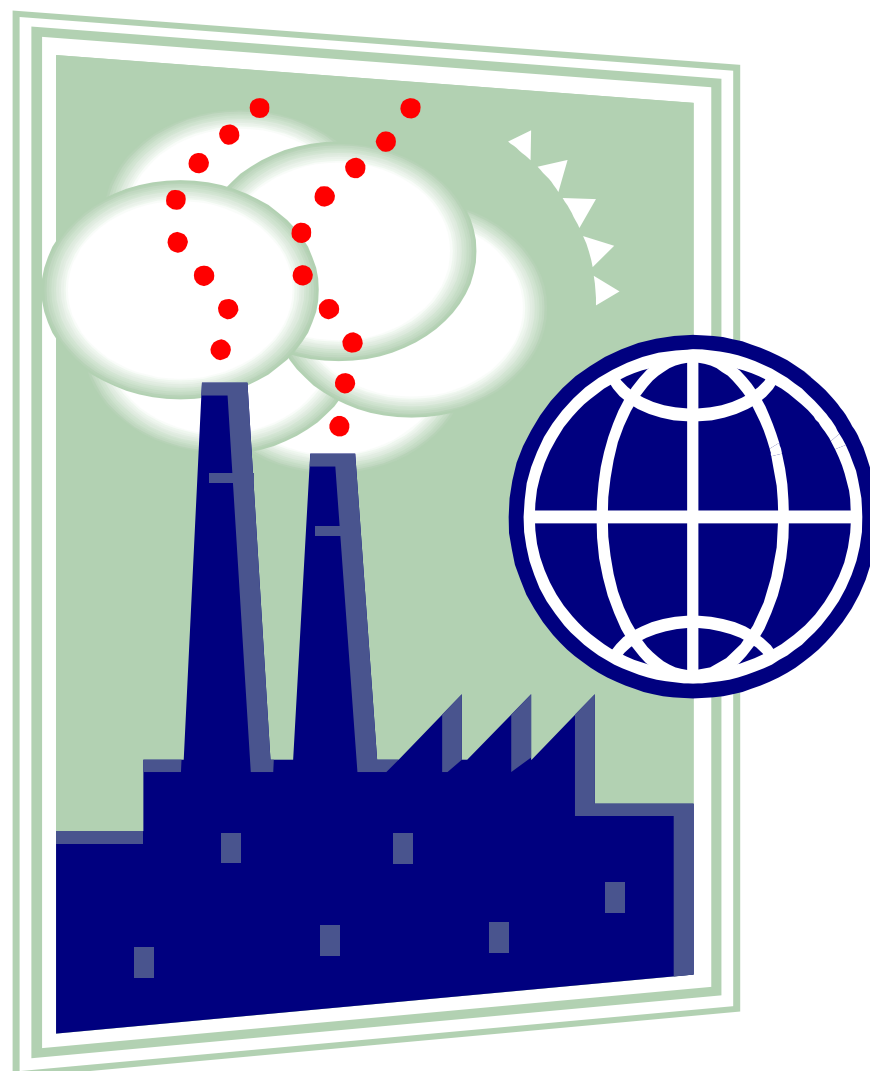


## The Bigger Energy Issue

- Environment
- Climate change
- CO<sub>2</sub> emissions
- Who is the biggest culprit?
  - Cars?
  - Planes?
  - No, actually, it's buildings



## World Energy Consumption

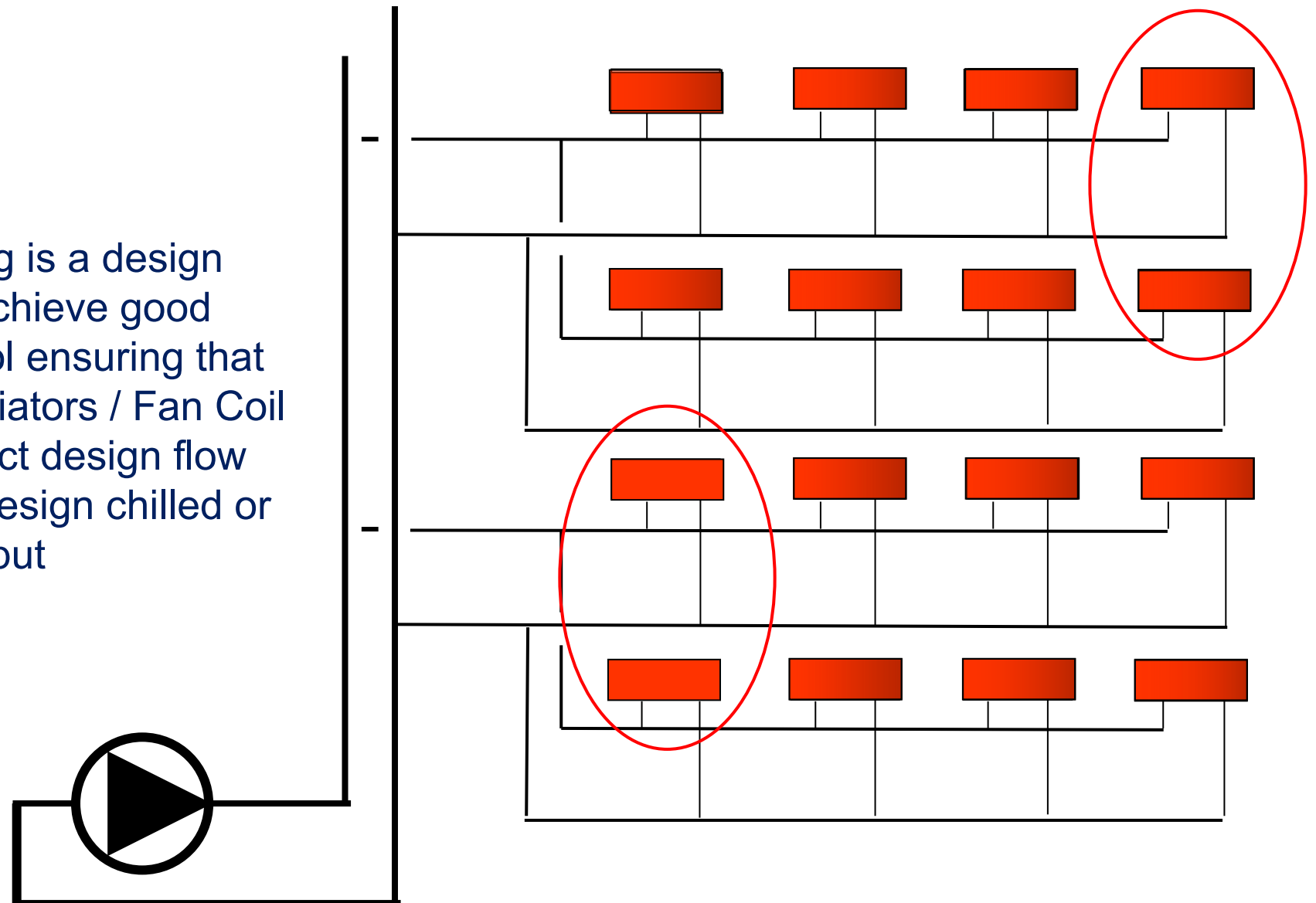


***40% of world energy consumption goes into buildings***

## Alleviating the Energy issue: Dynamic Balancing

Dynamic Balancing is a design methodology to achieve good environmental control ensuring that all terminal units (Radiators / Fan Coil Units etc.) get correct design flow rates and therefore design chilled or heat output

*unbalanced*  
*balanced*



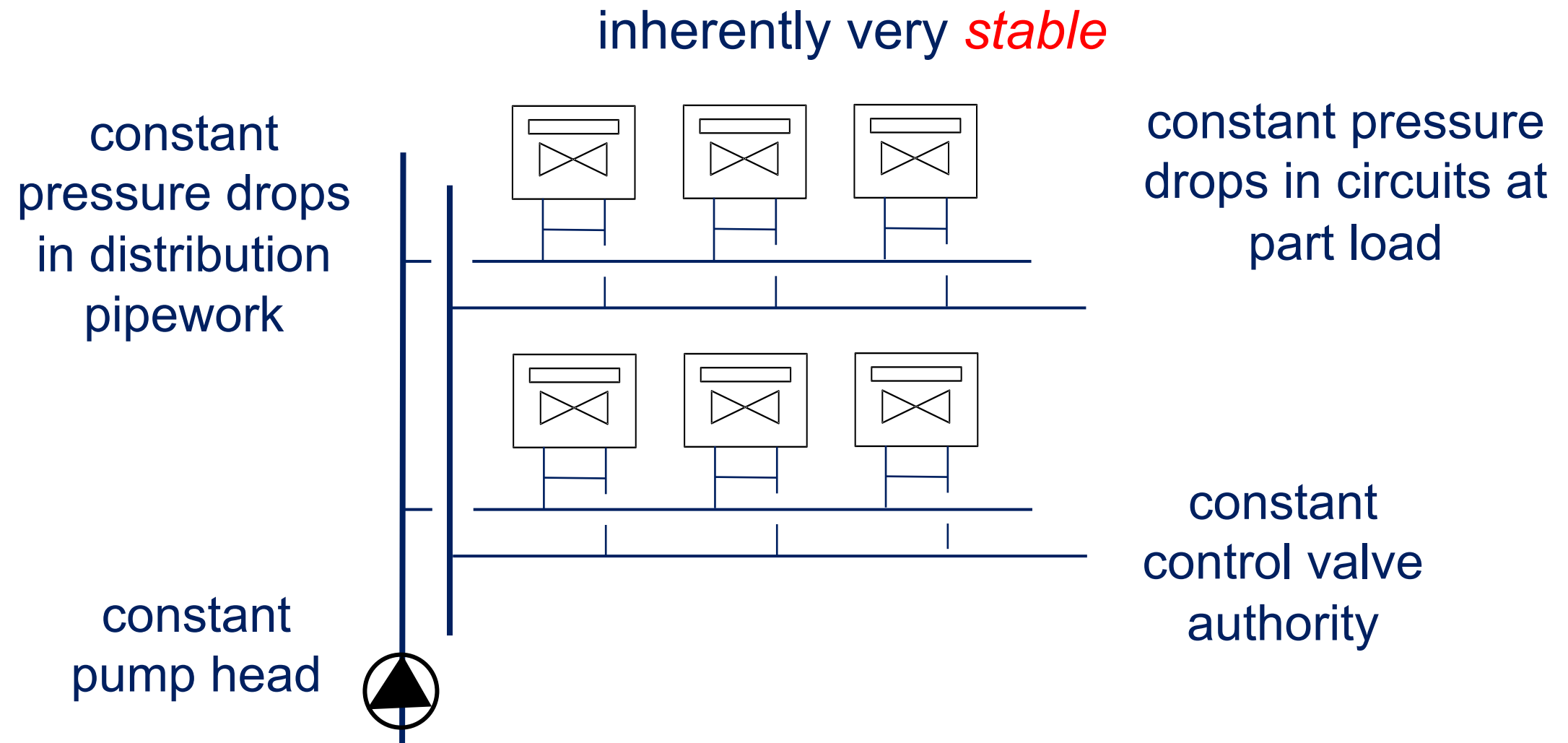
## Why do we need Dynamic Balancing?

Move from Constant Volume to Variable Volume systems

- driven by Government legislation
- energy conservation
- subsequent CO<sub>2</sub> emissions

The move from constant to variable flow design enables pump energy savings

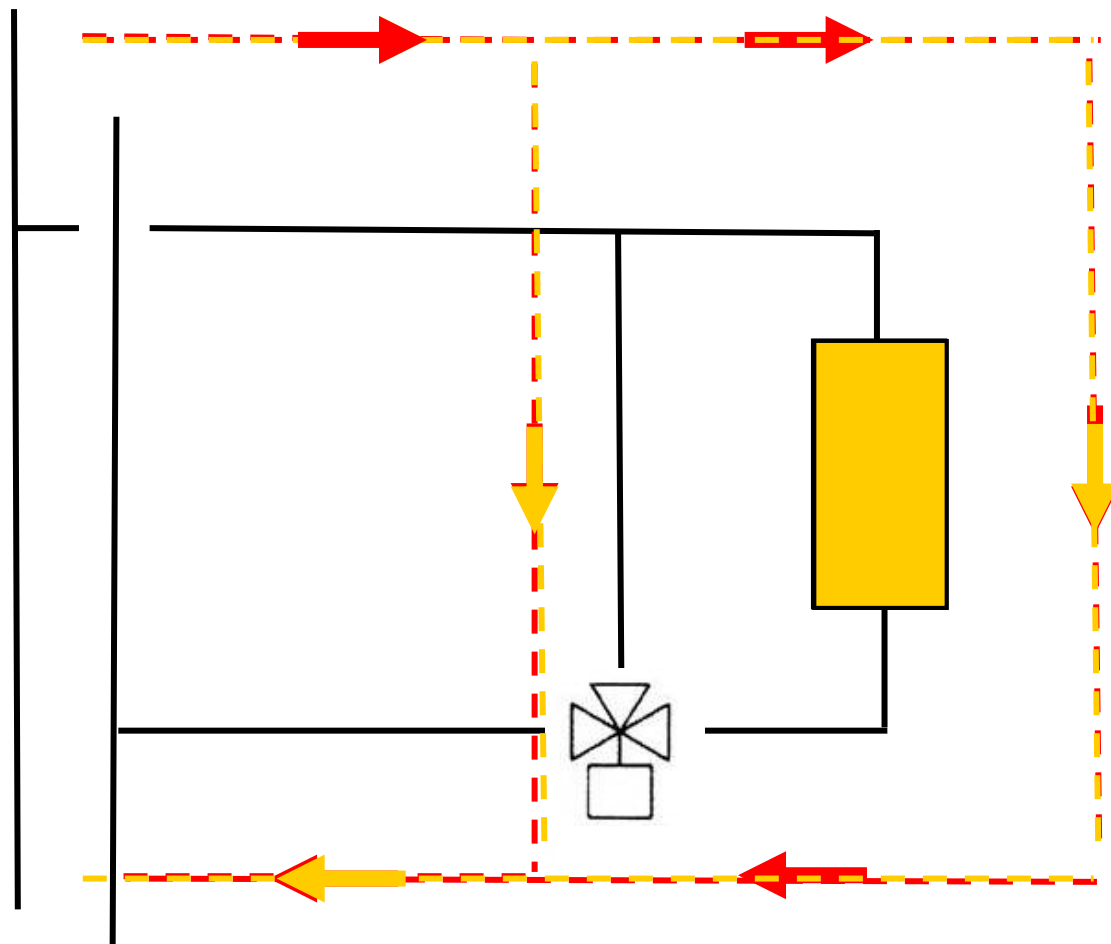
## Why do we need Dynamic Balancing – constant volume



***Control valves easy to size with constant authority***

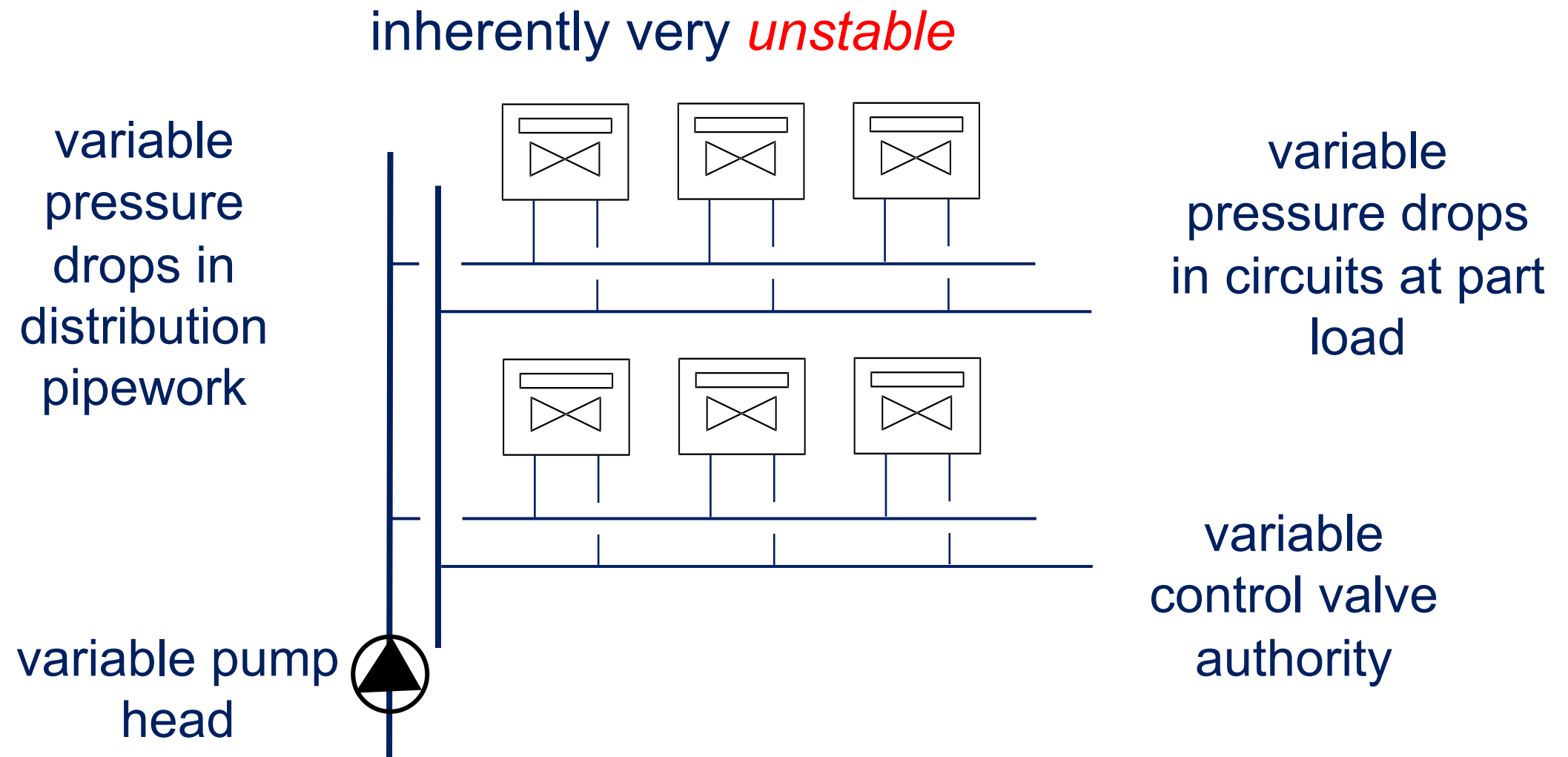


## Why do we need Dynamic Balancing – constant volume



- constant amount of water  
pumped around a system  
controlled by 3 or 4 port control  
valves and would be
- through terminal
  - split between terminal and  
by-pass
  - diverted back if not required

## Why do we need Dynamic Balancing – variable volume



***Control valves difficult to size with variable authority***



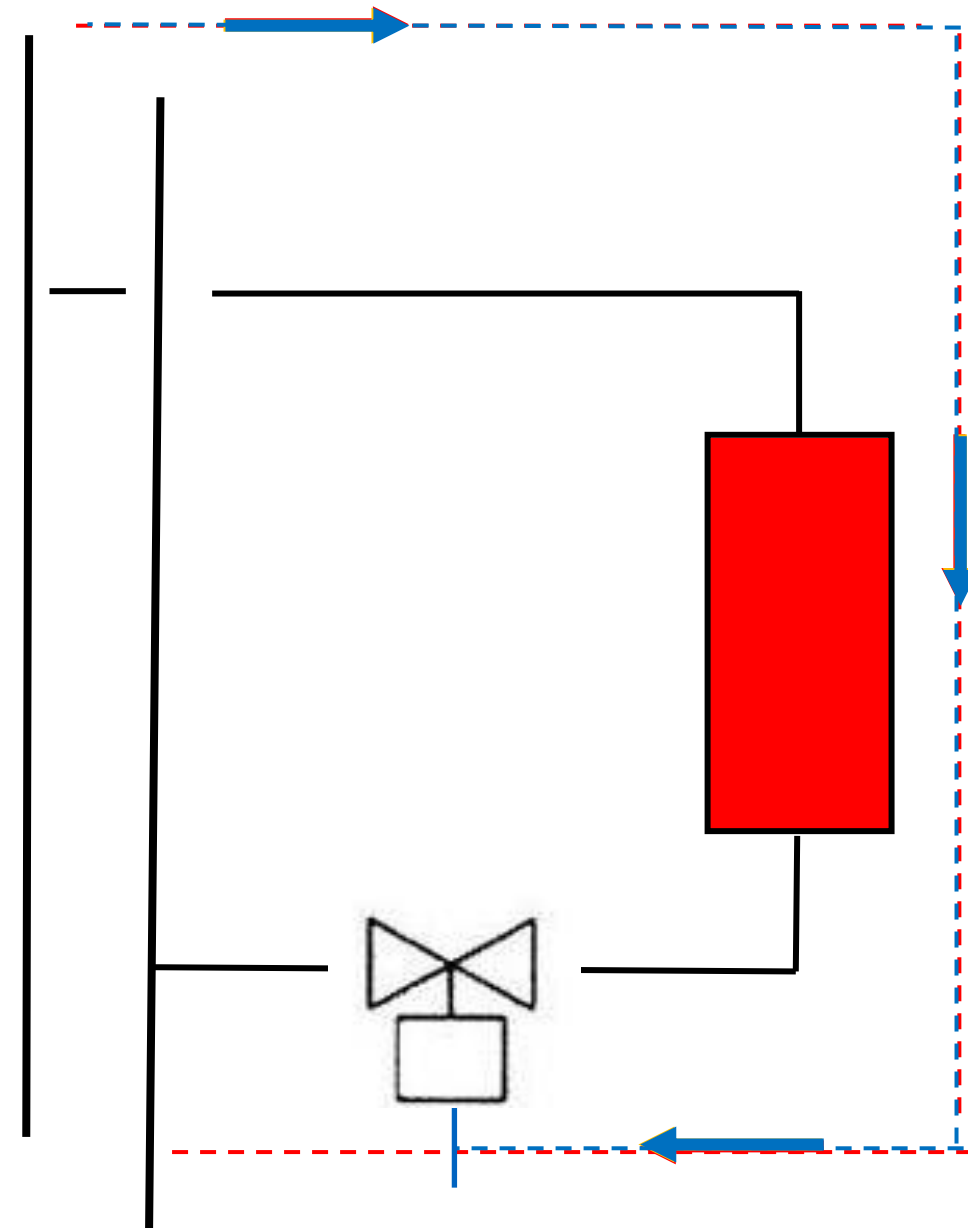
## Why do we need Dynamic Balancing – variable volume

Variable amount of water pumped around a system now controlled by 2 port control valves

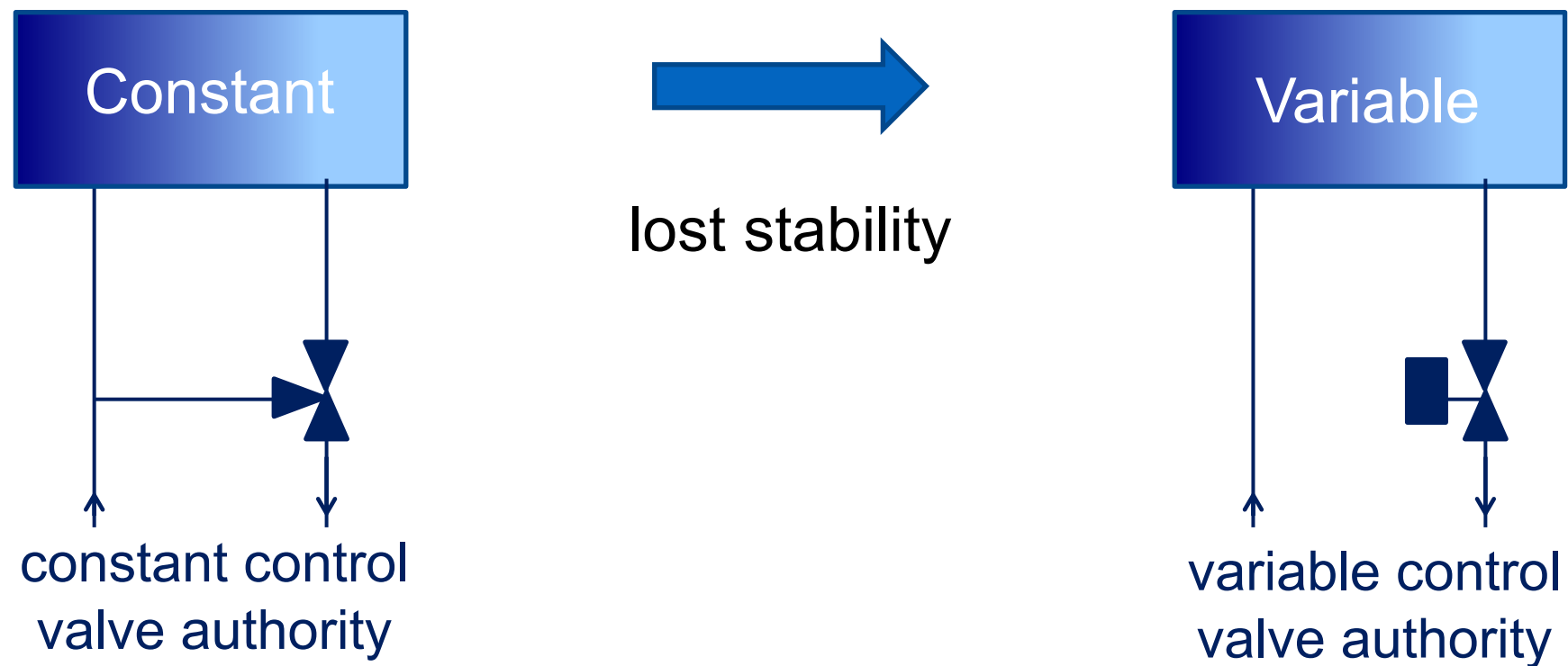
open

Modulating between open and closed

closed



## Why do we need Dynamic Balancing Valves

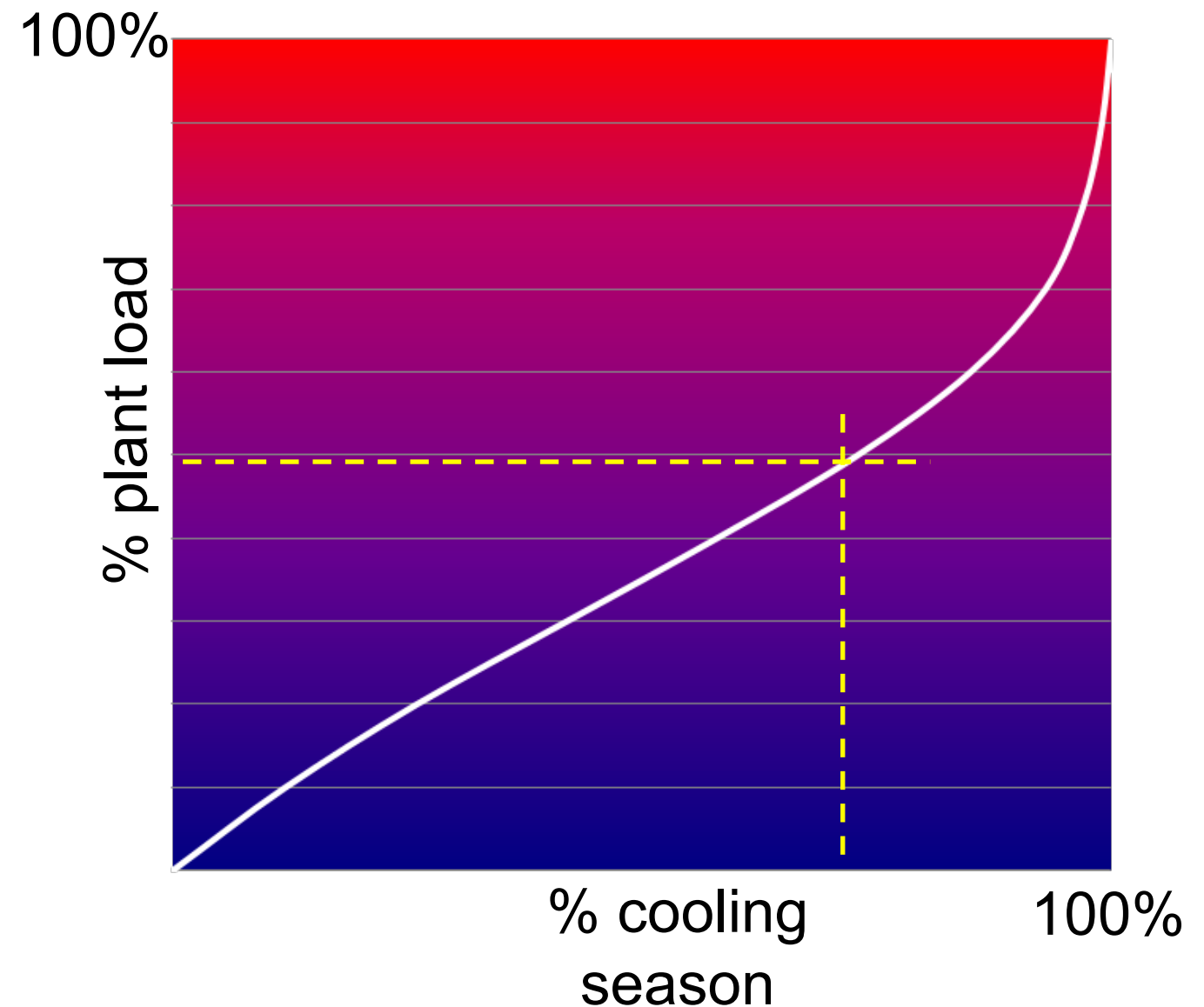


***DPCV installed to protect 2 port control valve authority  
or  
PICVs installed to replace commissioning / control valves***

## Why do we need Variable Volume Systems

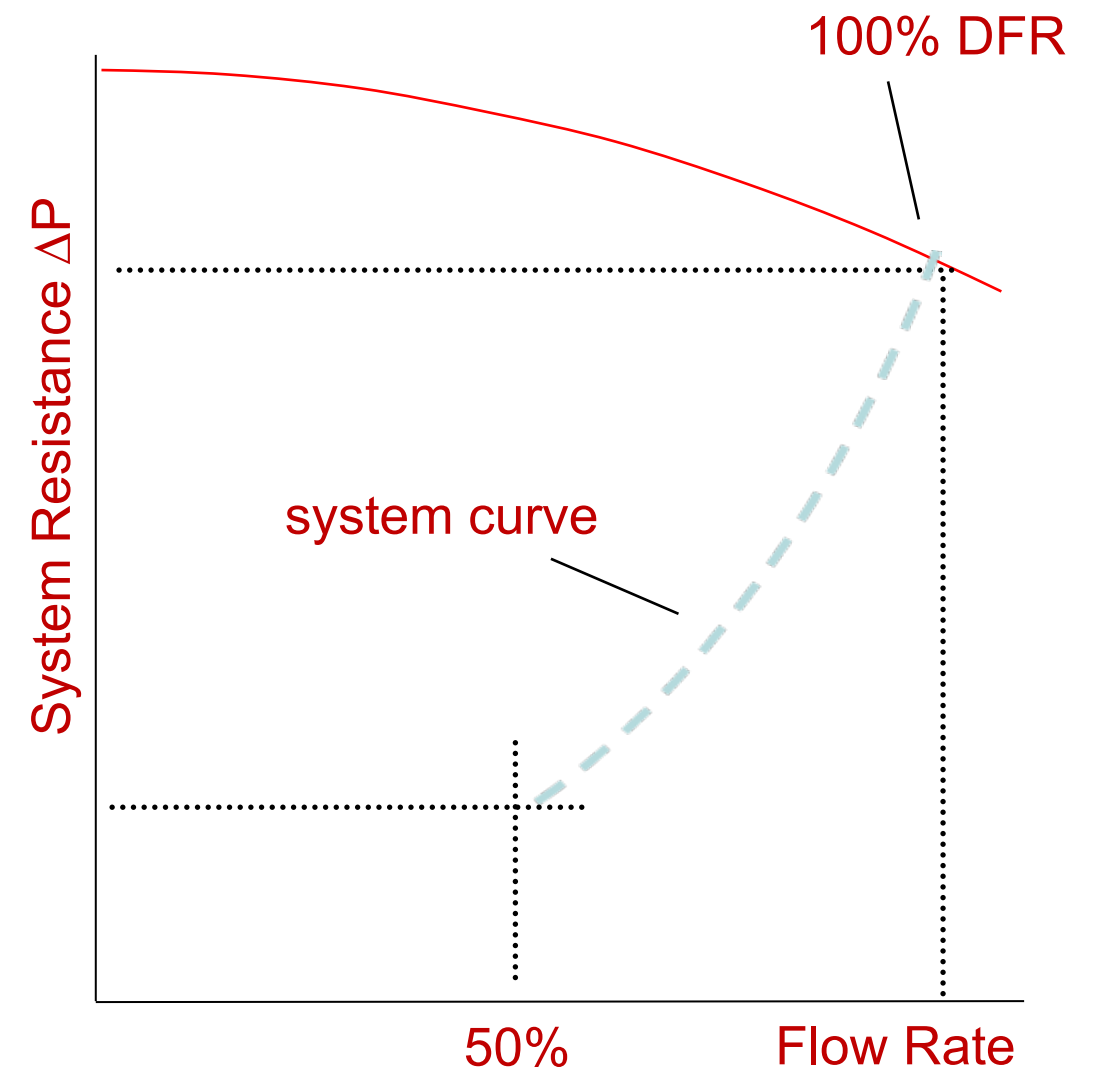
### Temperature variations

- Over 70% of cooling season
- Load is less than 50%



## Variable Volume Pump Energy Savings

- As system demand change, flow rate changes
- Direct relationship between pump speed and flow rate
- 50% pump speed = 50% flow rate



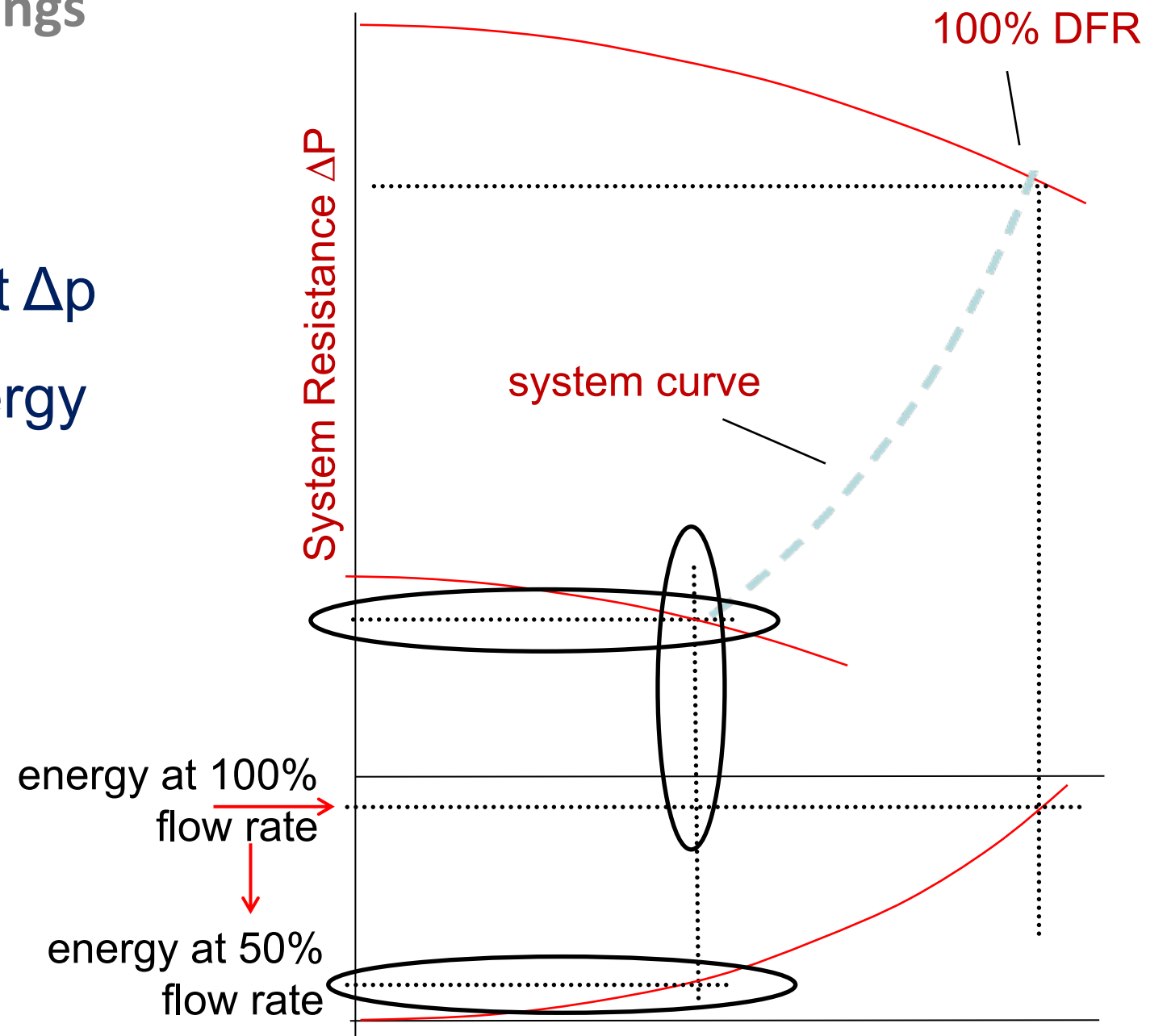
*50% flow rate = 25% System Resistance*

## Variable Volume Pump Energy Savings

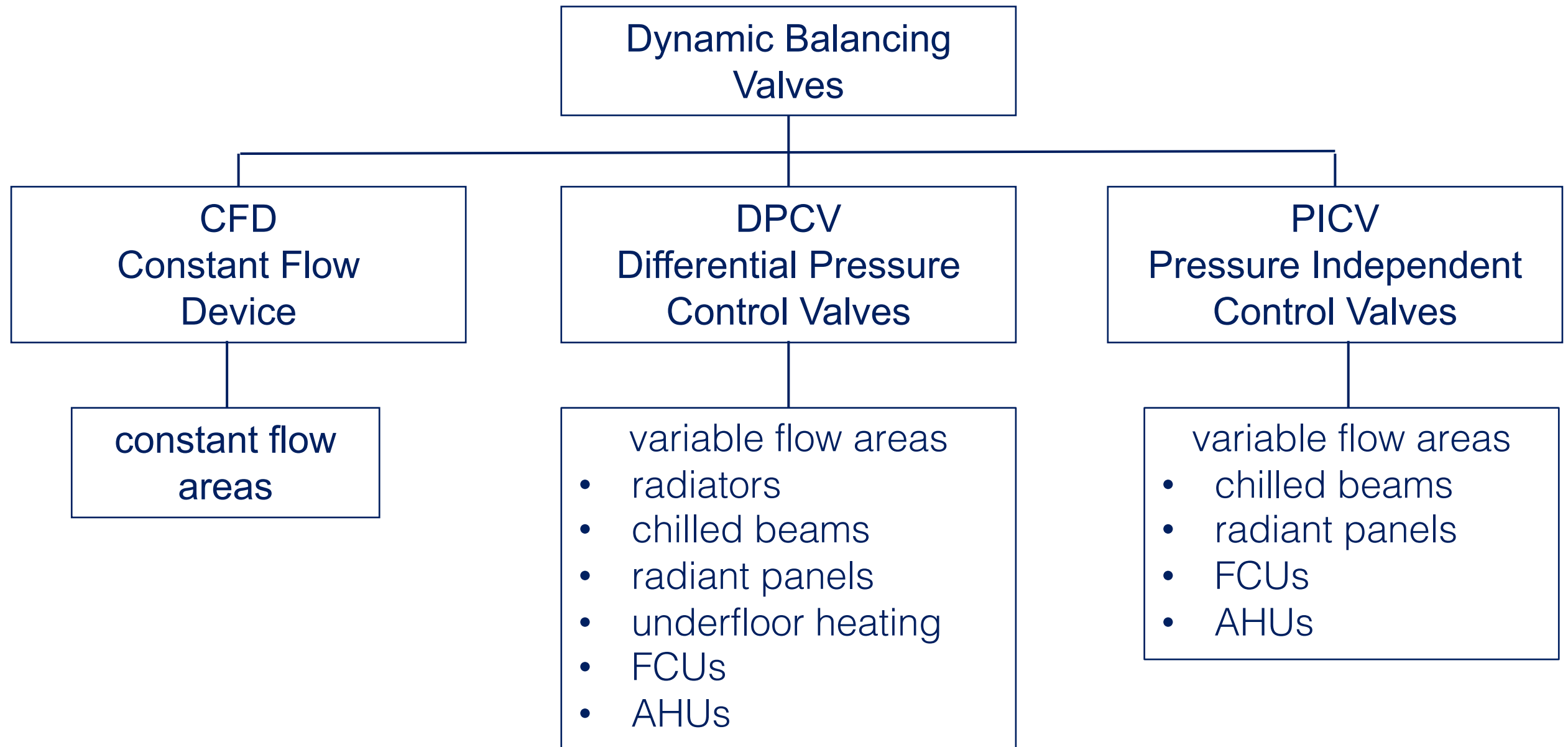
- 50% speed = 50% flow rate
- 50% flow rate = 25% circuit  $\Delta p$
- 25% circuit  $\Delta p$  = 12.5% energy

*50% flow rate = over 70%  
pump energy saving*

*Loses due to pump  
efficiency reducing*



## What are Dynamic Balancing Valves?



## What are Dynamic Balancing Valves?

Dynamic Balancing  
Valves

CFD  
Constant Flow  
Device

Constant flow  
areas





## What are Dynamic Balancing Valves?

Dynamic Balancing  
Valves

DPCV  
Differential Pressure  
Control Valves

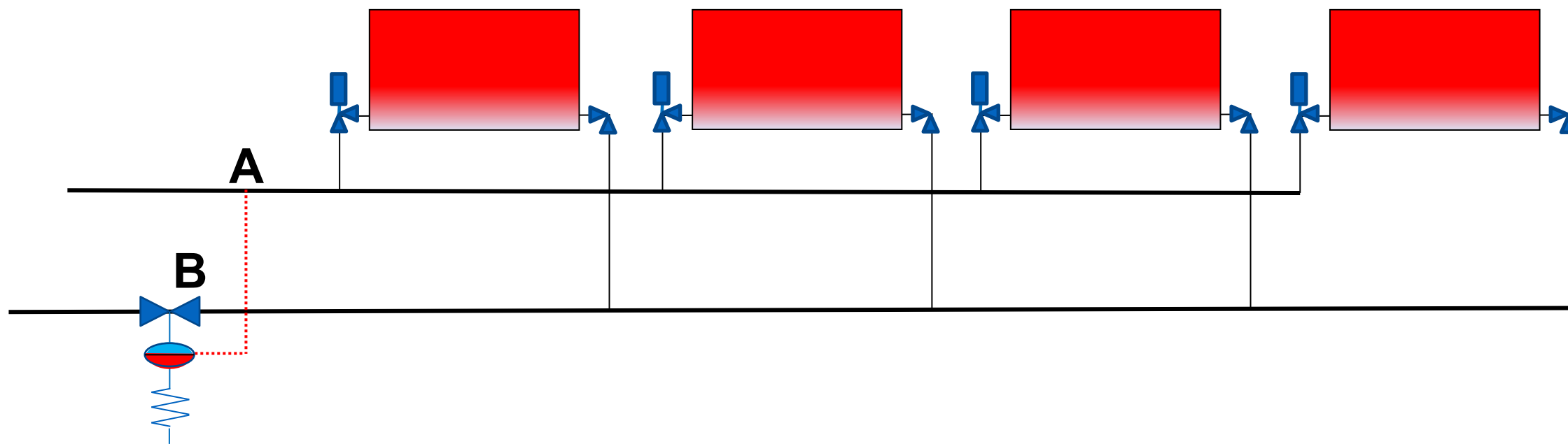
Variable flow areas

- Radiators
- Chilled beams
- Radiant panels
- Underfloor heating
- FCUs
- AHUs



## Why do we need Dynamic Balancing – radiator circuits

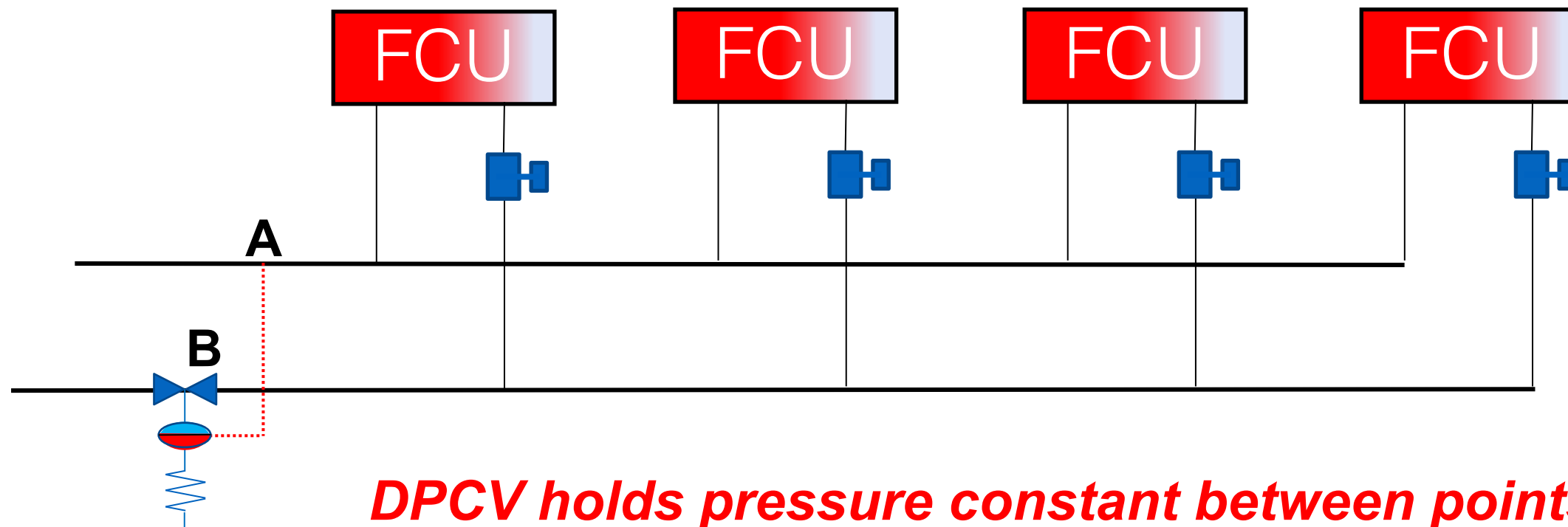
- To enable TRV (Thermostatic Radiator Valves) to correctly operate, a DPCV is installed to limit the radiator sub-circuit pressure differential
- The installation of DPCVs in sub-circuits with TRVs reduces the pressure that the TRV has to close against thus reducing the possibility of noisy valves



***DPCV holds pressure constant between points A and B***

## Why do we need Dynamic Balancing – DPCVs for FCU

- To enable modulating 2 port control valves to operate with an acceptable authority, a DPCV is installed to limit the circuit pressure differential
- The installation of DPCVs on sub-branches with 2 port control valves is therefore essential to achieve good control



## Why do we need DPCVs - control valve authority

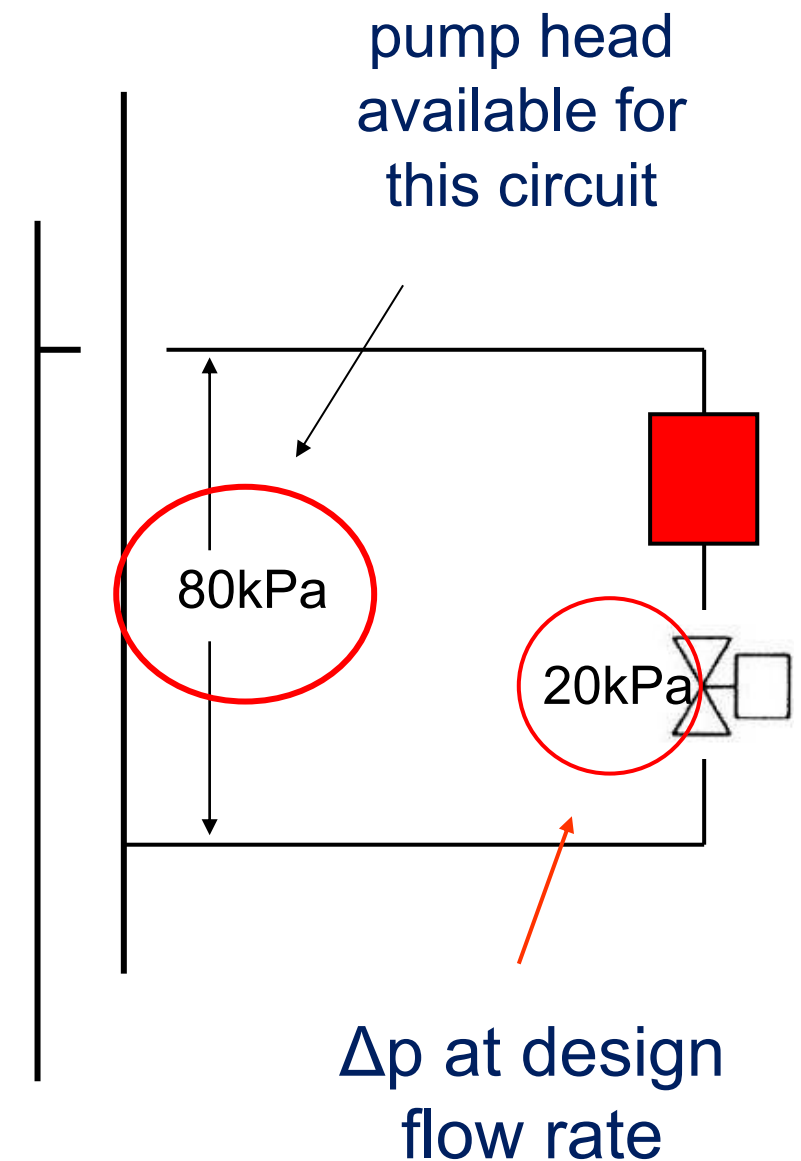
### Example without DPCV

$$\text{valve authority } \beta = \frac{\Delta p \text{ across 2 port}}{\Delta p \text{ across circuit}}$$

$$\beta = \frac{20 \text{ kPa}}{80 \text{ kPa}}$$

$$\beta = 0.25$$

**Too low - unacceptable**



## Why do we need DPCVs - control valve authority

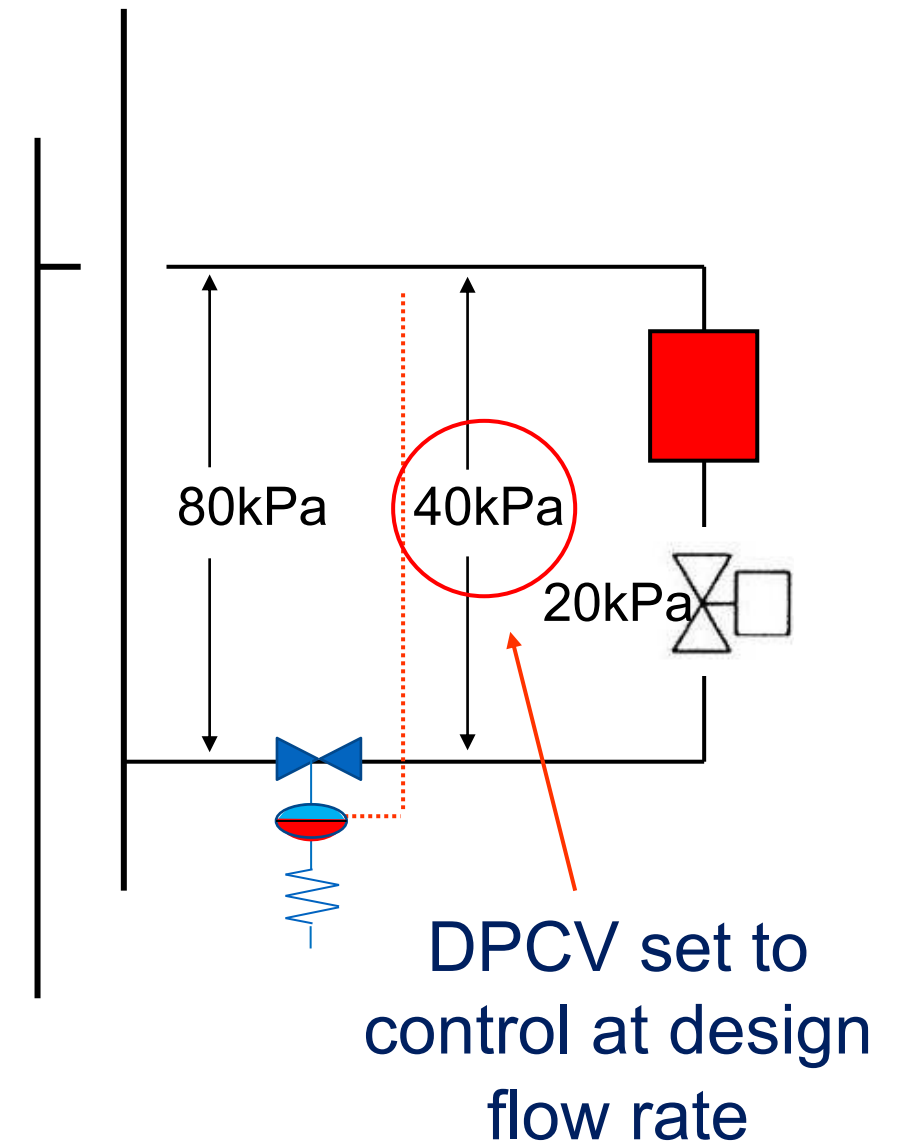
Example with DPCV fitted

$$\text{Valve authority } \beta = \frac{\Delta p \text{ across 2 port}}{\Delta p \text{ across circuit}}$$

$$\beta = \frac{20 \text{ kPa}}{40 \text{ kPa}}$$

$$\beta = 0.5$$

**Acceptable**



## Why do we need DPCVs - control valve authority

### Position of DPCV?

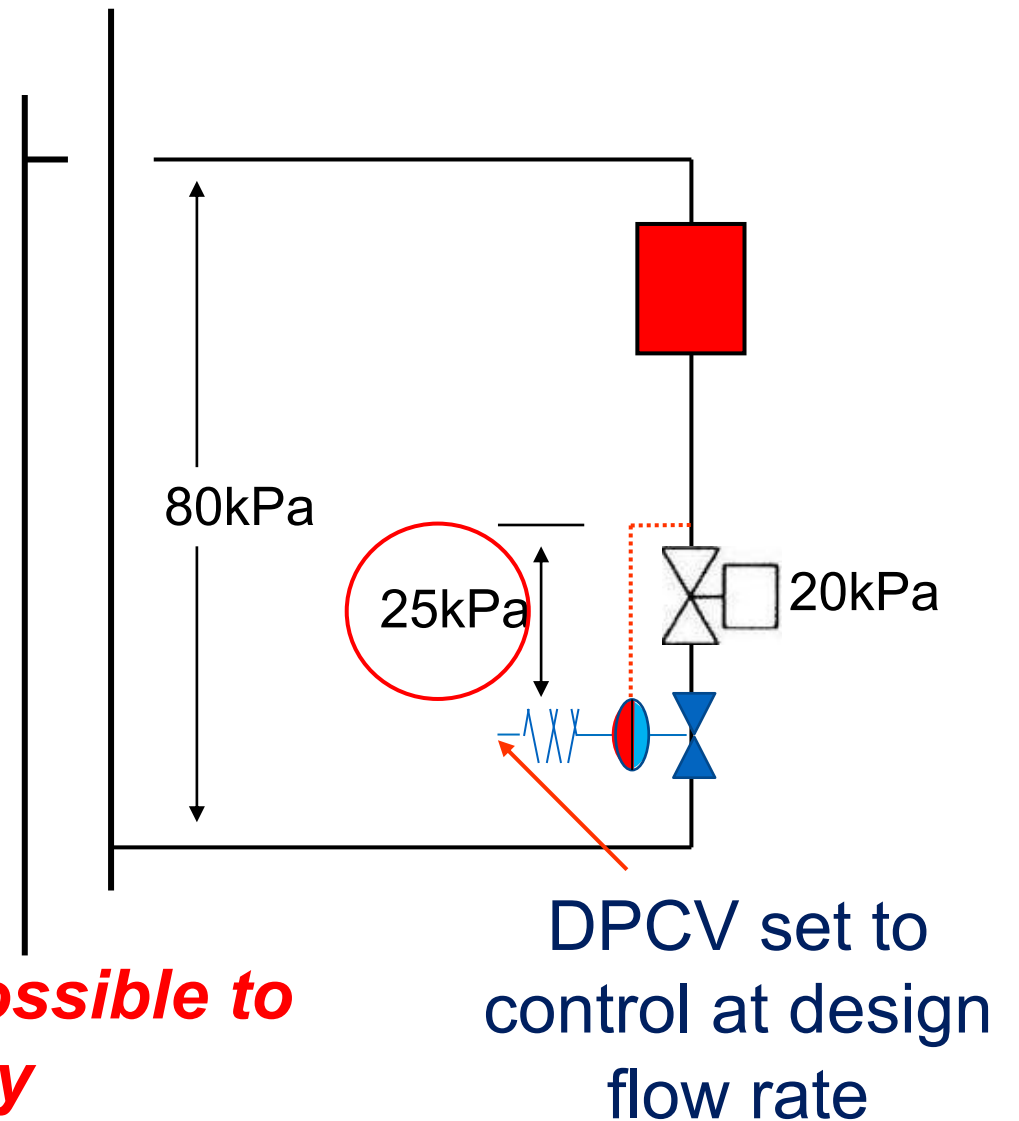
$$\text{valve authority } \beta = \frac{\Delta p \text{ across 2 port}}{\Delta p \text{ across circuit}}$$

$$\beta = \frac{20 \text{ kPa}}{25 \text{ kPa}}$$

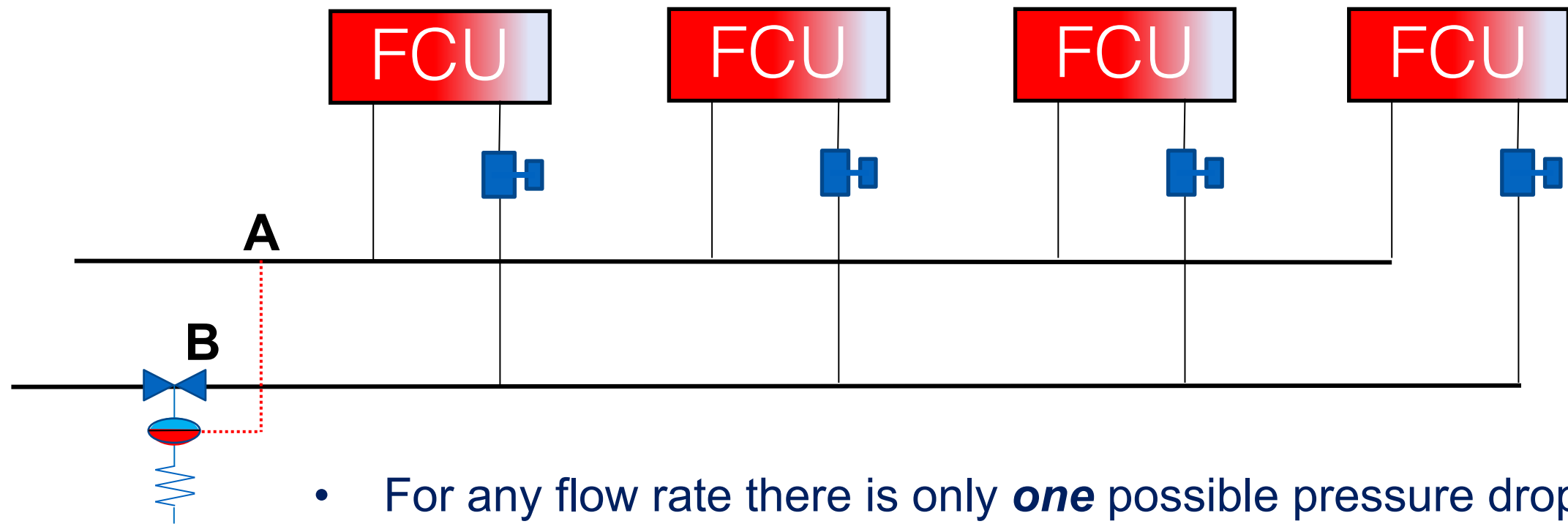
$$\beta = 0.8$$

*position can influence authority*

*on single terminal circuits – as closes as possible to control valves gives higher authority*



### How do DPCVs work

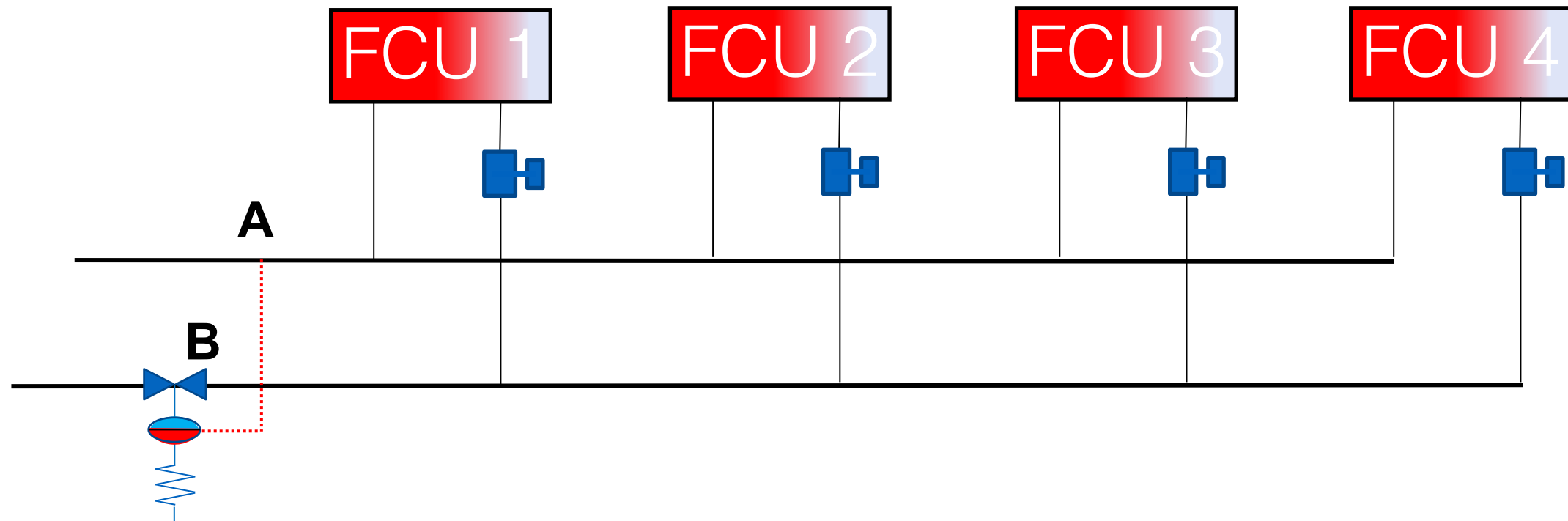


- For any flow rate there is only **one** possible pressure drop between any 2 points
- As flow rate changes  $\Delta p$  changes – squared change  
10% in flow = 21% in  $\Delta p$

***DPCV holds pressure constant between points A and B***



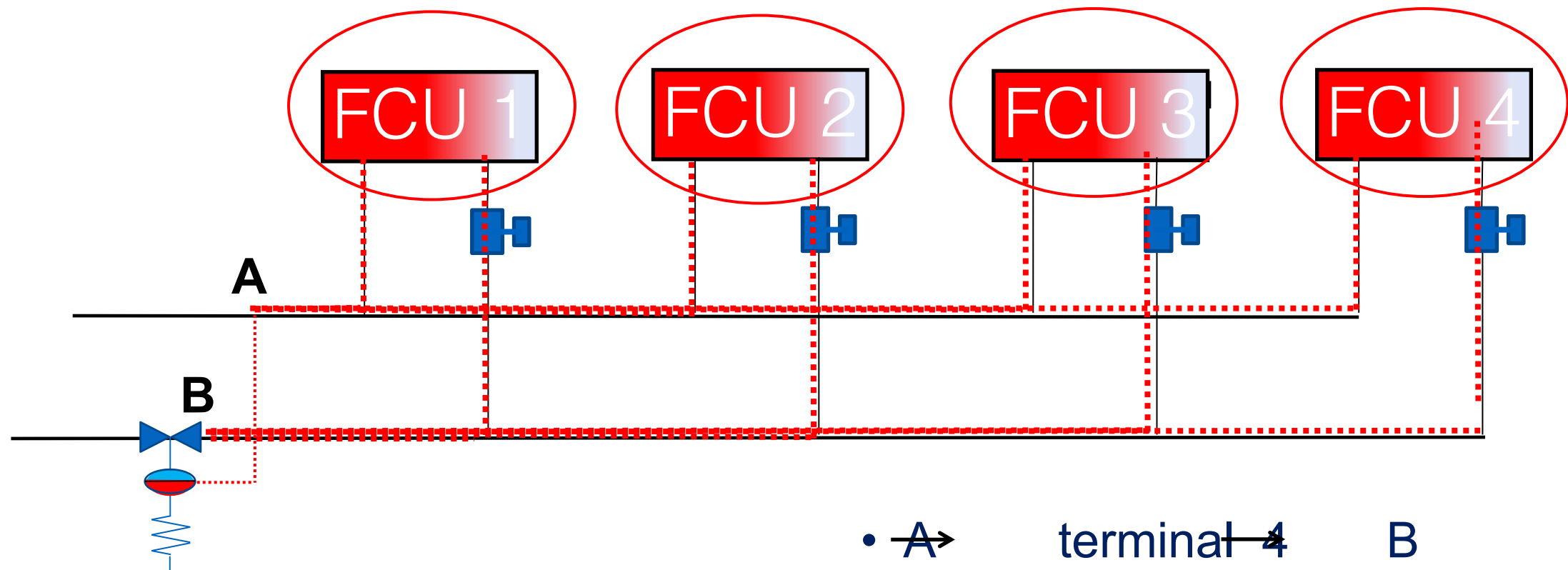
### Why a DPCVs work



Kirchhoff's 2nd law states

*'Head loss in parallel circuits must be equal'*

### Why a DPCVs work



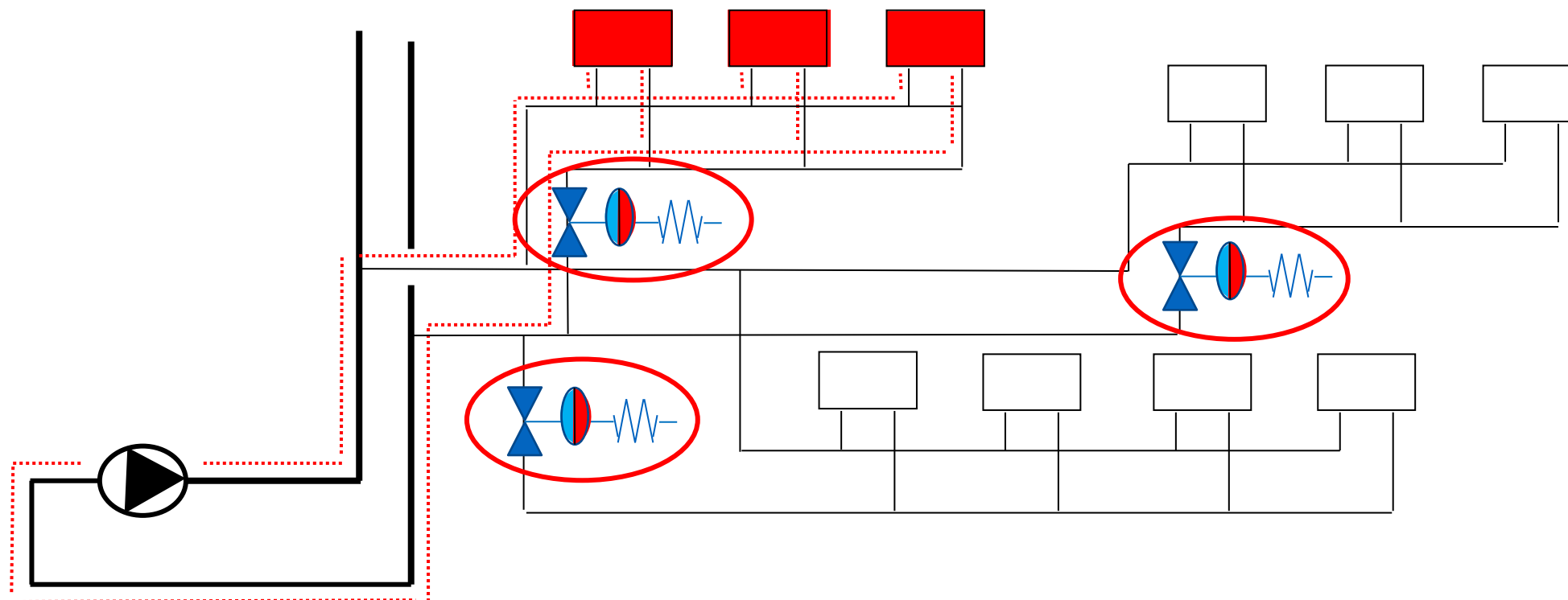
Applying Kirchhoff's law

- $\overrightarrow{A}$  terminal  $\rightarrow 4$  B
- $\overrightarrow{A}$  terminal  $\rightarrow 3$  B
- $\overrightarrow{A}$  terminal  $\rightarrow 2$  B
- $\overrightarrow{A}$  terminal  $\rightarrow 1$  B

***All have the same pressure drop irrespective of flow***

## Where do DPCVs go

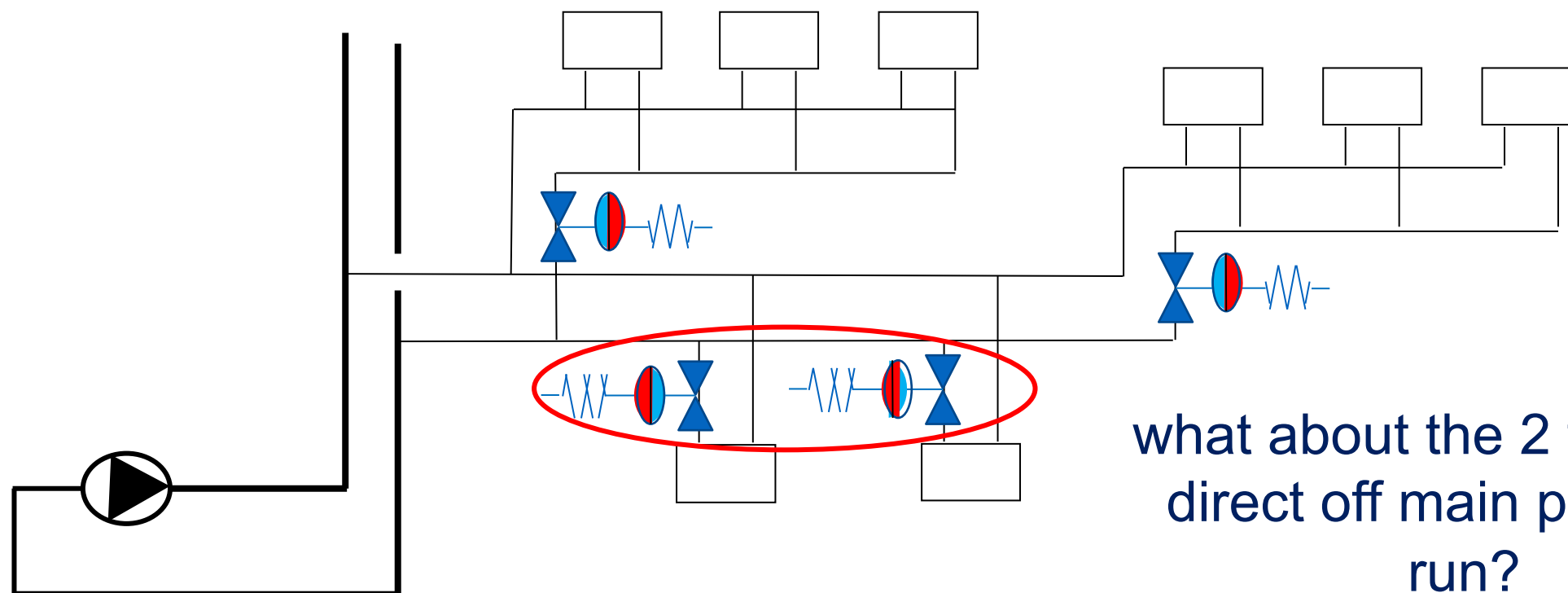
Branches are broken down into sub-circuits, each controlled by a DPCV



***Each circuit must only flow through a single DPCV***

## Where do DPCVs go

Branches are broken down into sub-circuits, each controlled by a DPCV

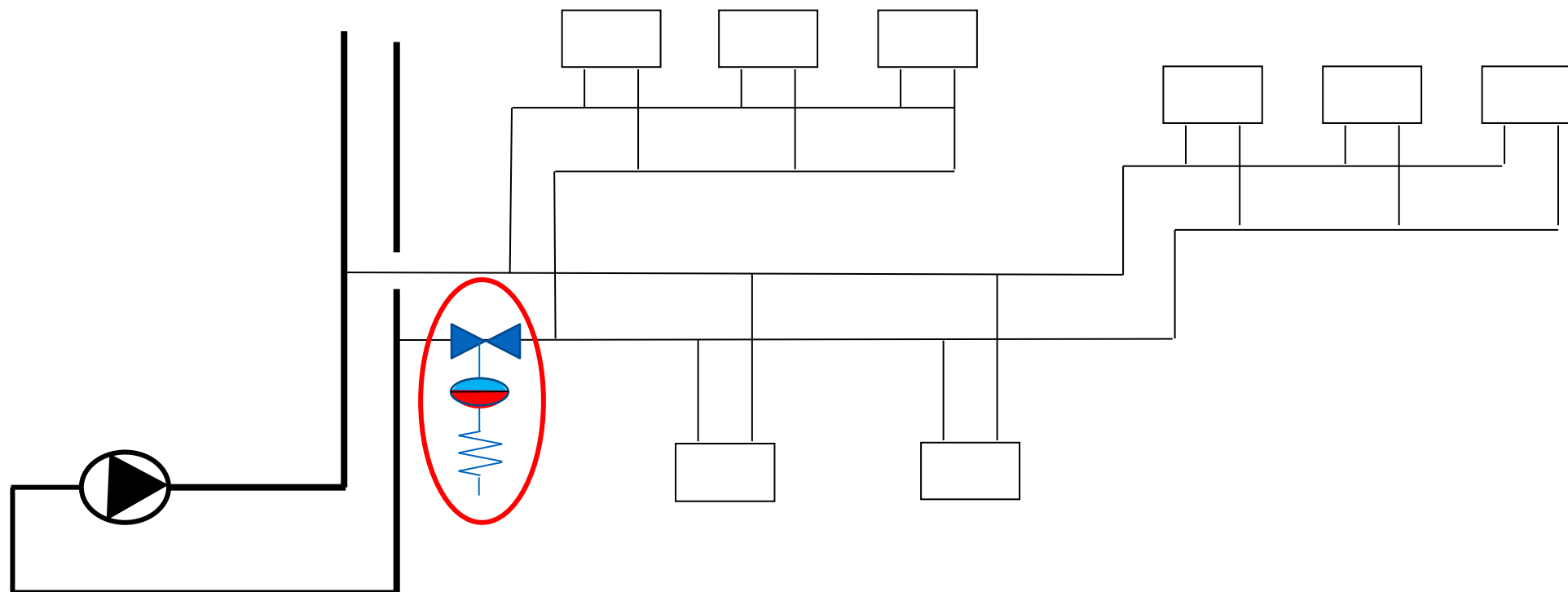


what about the 2 terminals  
direct off main pipework  
run?

***Each circuit must have some protected from over flow***

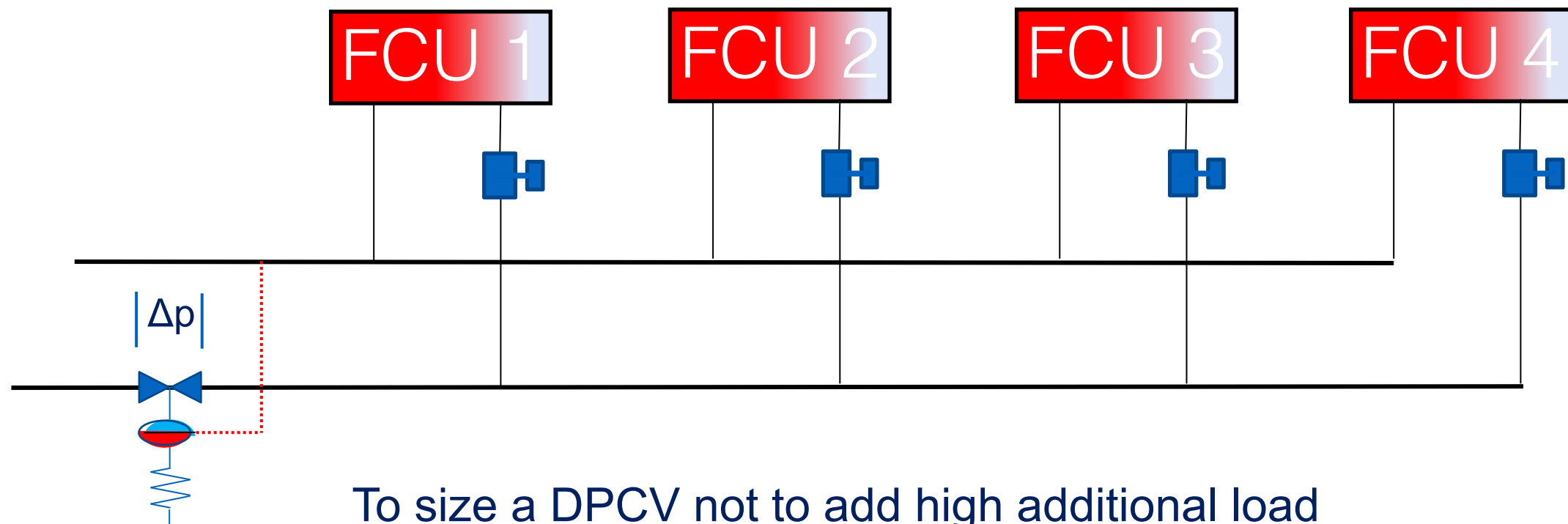
Where do DPCVs go

Alternative approach



***Complete circuit protected by a single DPCV***

### How are DPCVs selected



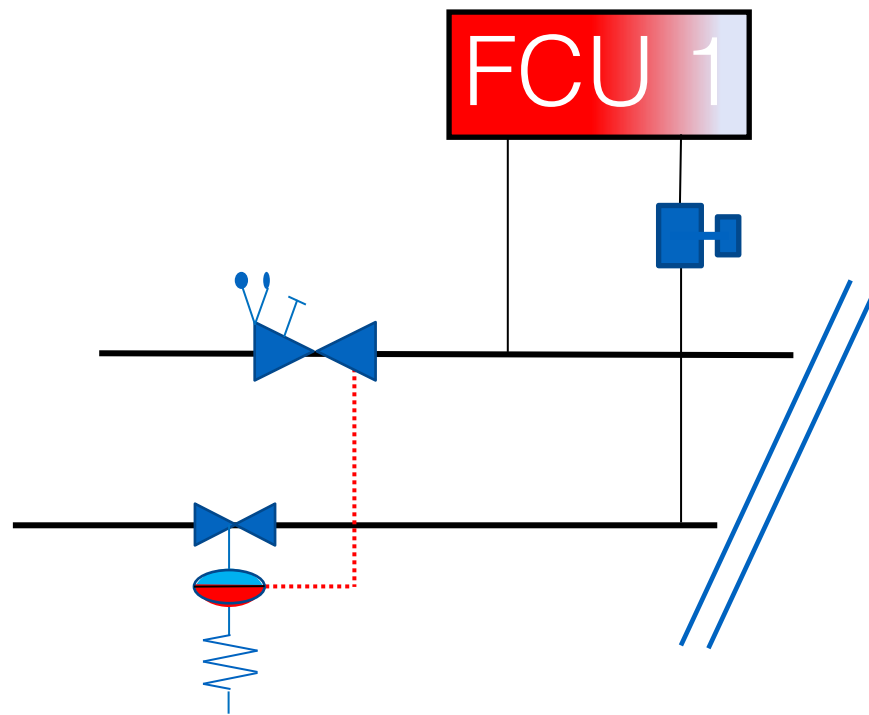
To size a DPCV not to add high additional load

the  $\Delta p$  across the DPCV

*is typically 10 - 20kPa*

irrespective of the  $\Delta p$  through the circuit

### Other Valves associated with DPCVs - Companion Valve



#### Companion Valve offers

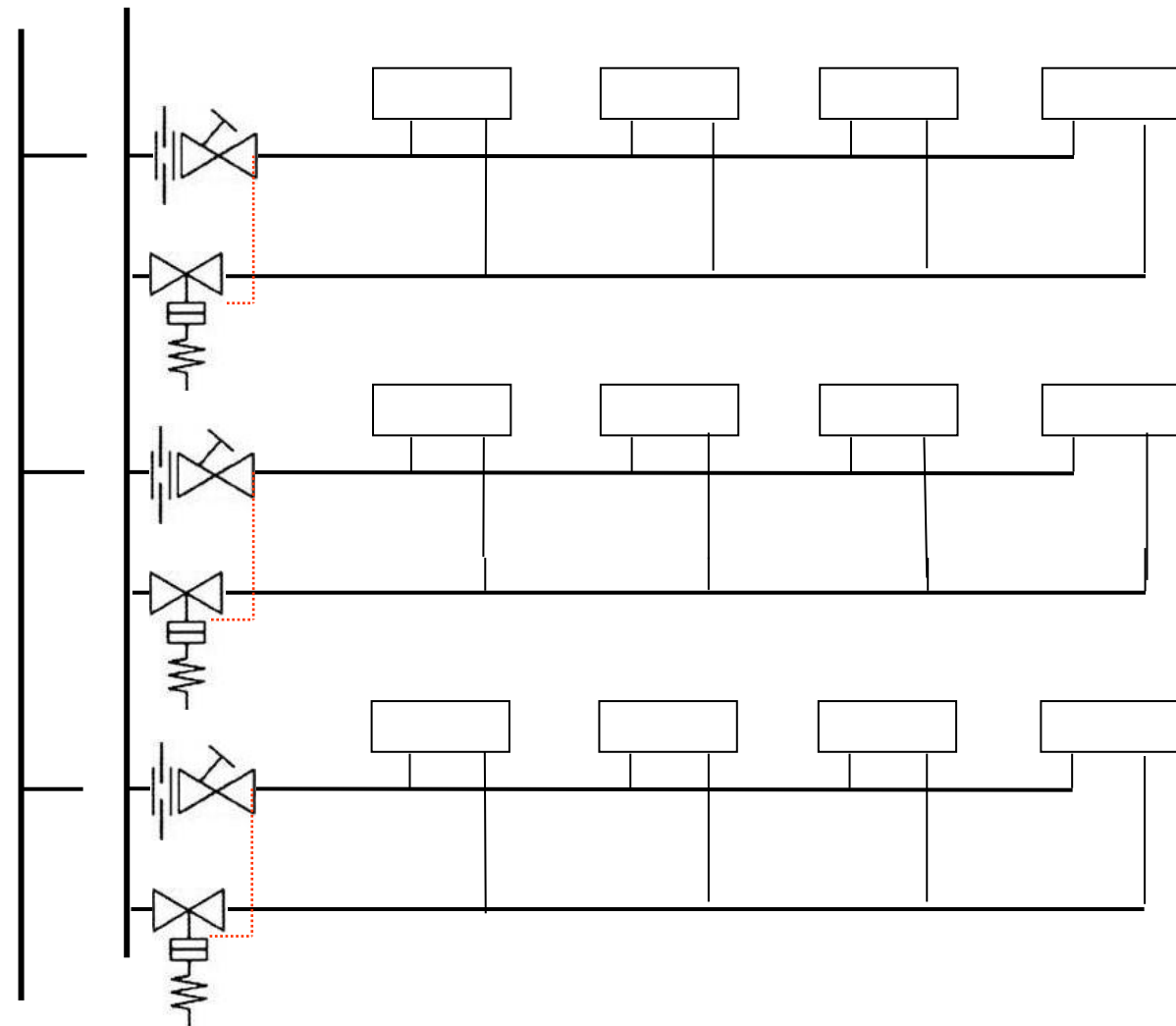
- flow measurement
- isolation
- DPCV connection
- bosses for test points





### Dynamic Balancing of the System

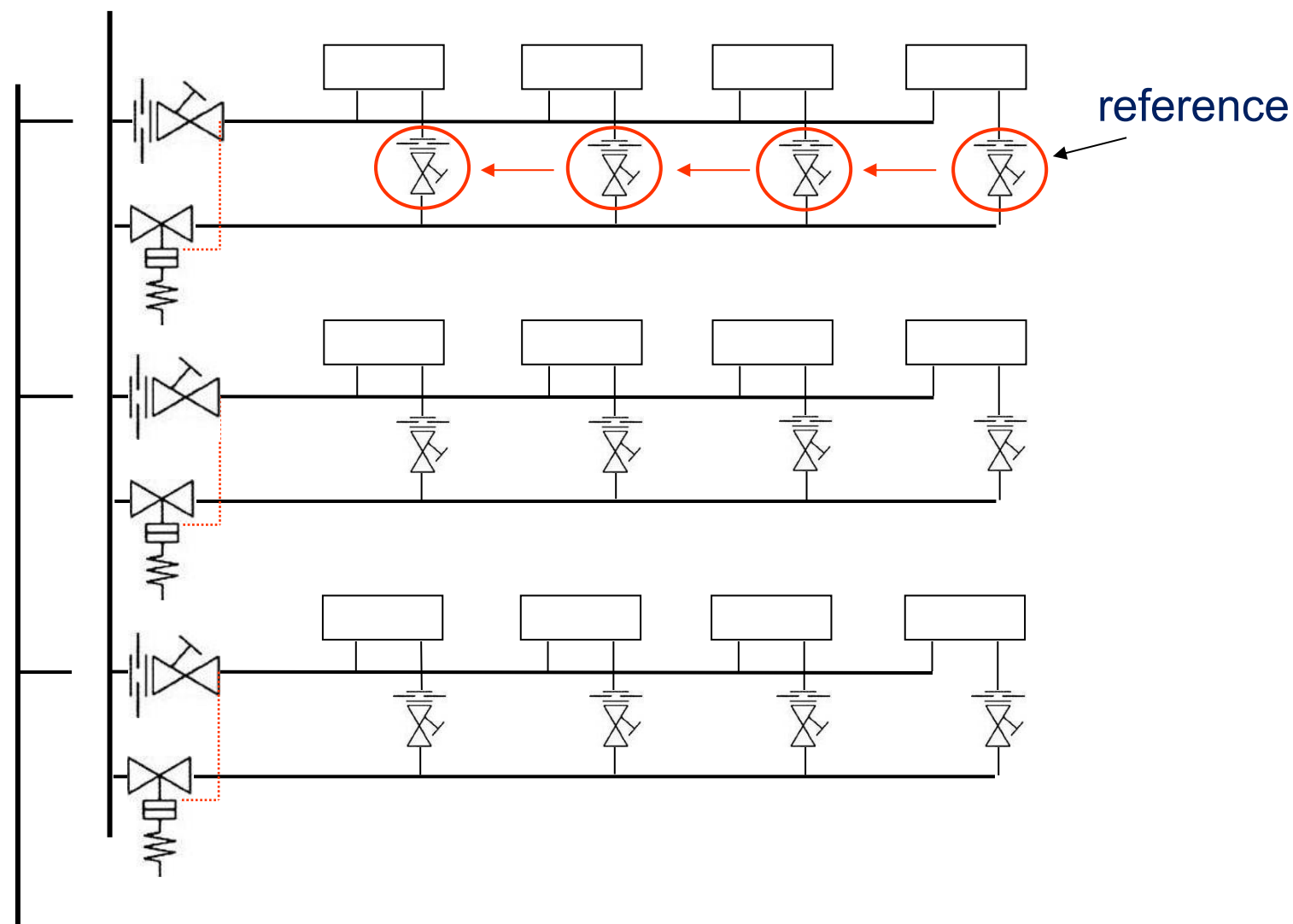
Because each sub-circuit is separated by a DPCV from fluctuating system pressure & therefore holds a constant pressure within the sub-circuit, commissioning sub-circuits can be carried out totally independently



***sub-circuits are independent of each other***

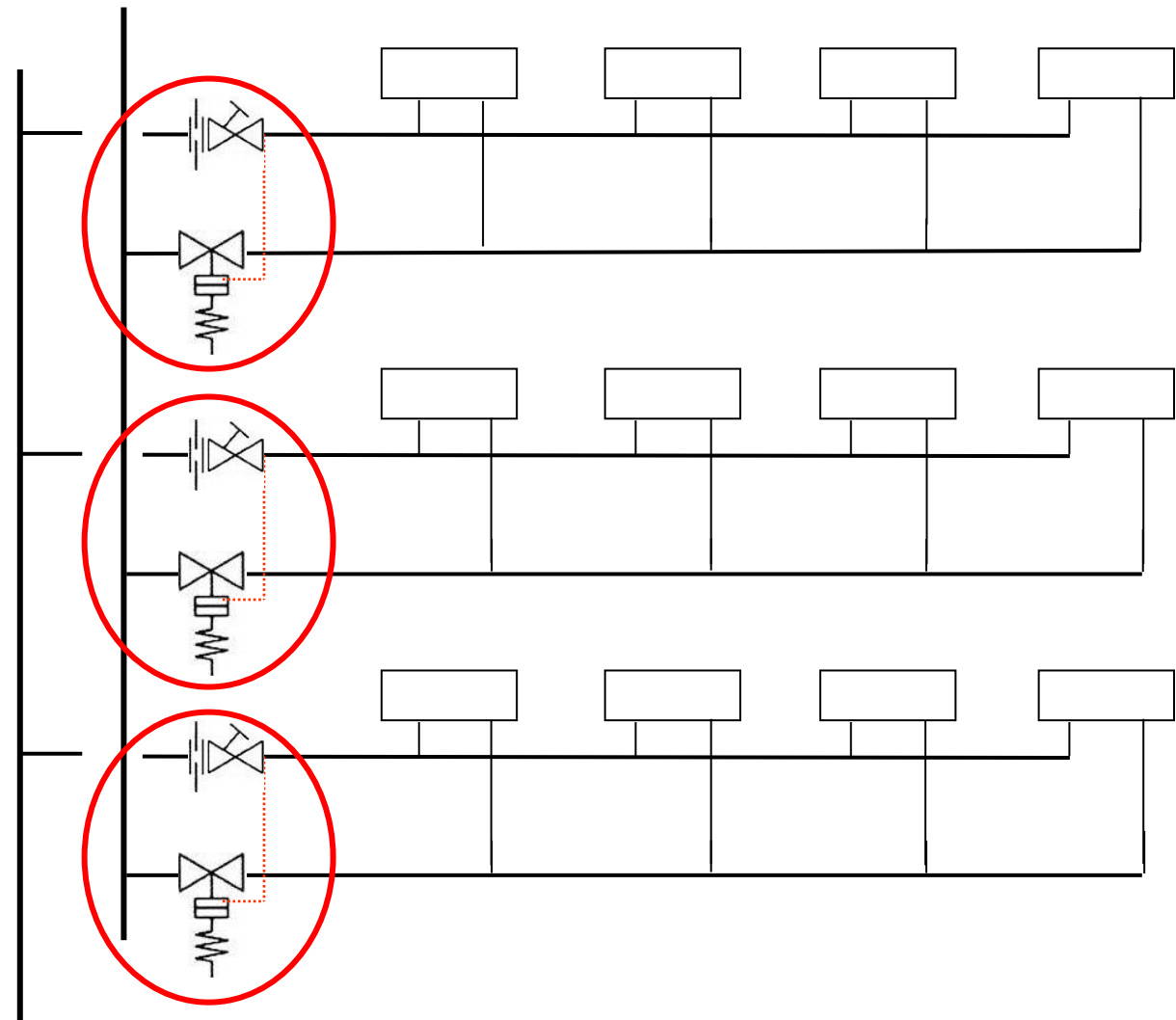
## Dynamic Balancing of the System

Commissioning within the sub-circuits is carried out by 'proportional balancing' in the conventional manner



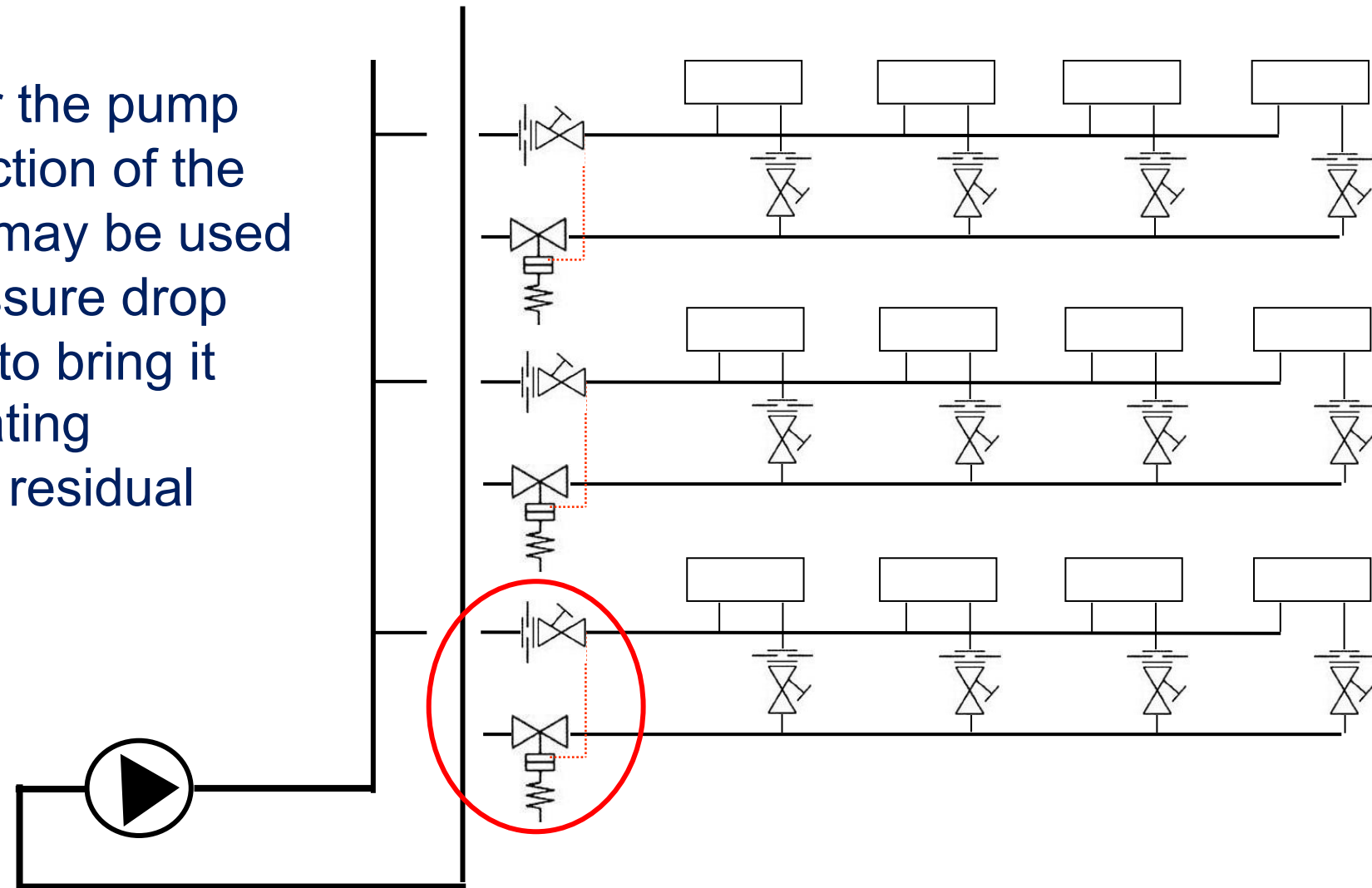
## Dynamic Balancing of the System

Each sub-circuit is balanced by measuring flow thro the 'Companion Valve' and adjusting DPCV to regulate flow

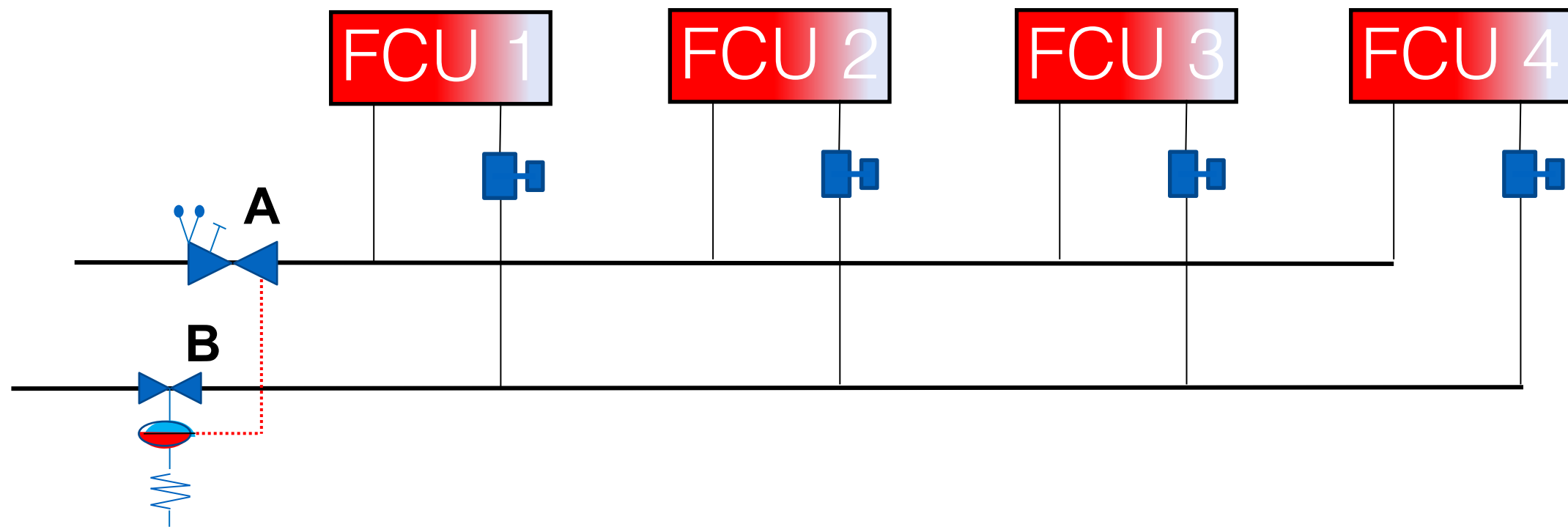


## Dynamic Balancing of the System

For circuits nearer the pump the regulating function of the companion valve may be used to reduce the pressure drop across the DPCV to bring it into a better operating position, i.e. splits residual pressure



### Dynamic Balancing of the System



1. set DPCV to highest control range setting
  - highest pressure difference gives highest flow rates
2. measure flow rate at companion valve
3. adjust DPCV until design flow rate achieved

## Variable Volume System

At maximum pump turndown, typically 10 - 20%, consideration needs to be given to branches to ensure

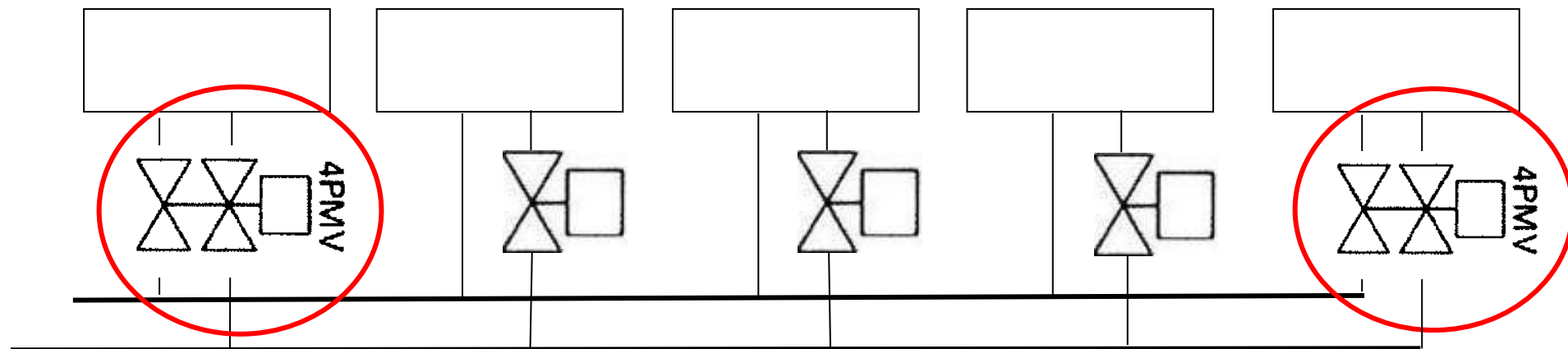
- pump flow at minimum load
- circulation of water treatment
- ready supply of heating / chilled water

## Dynamic Balancing of the System

possible solution

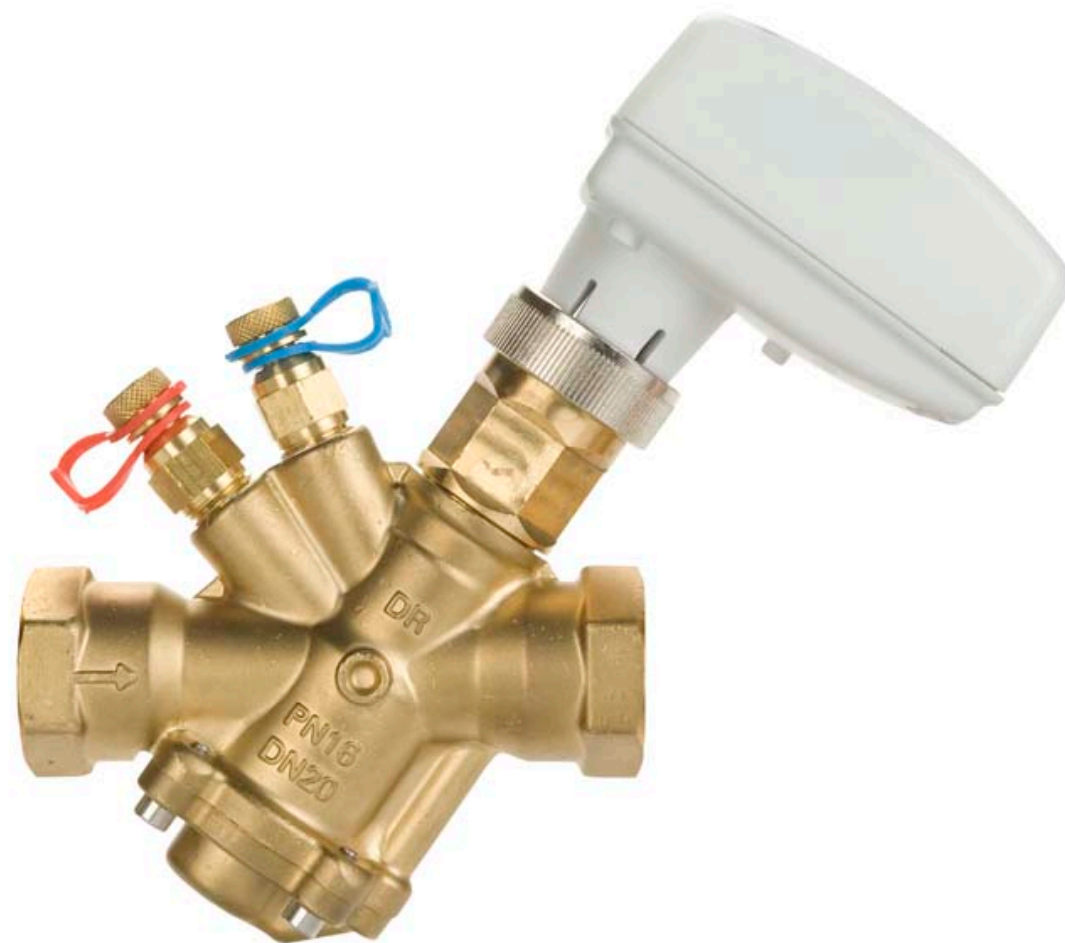
end terminal could have a 3 or 4 port control valve

➤ on larger circuits additional 3 or 4 ports could be added





## What are Dynamic Balancing Valves?



Dynamic Balancing  
Valves

**PICV**  
Pressure Independent  
Control Valves

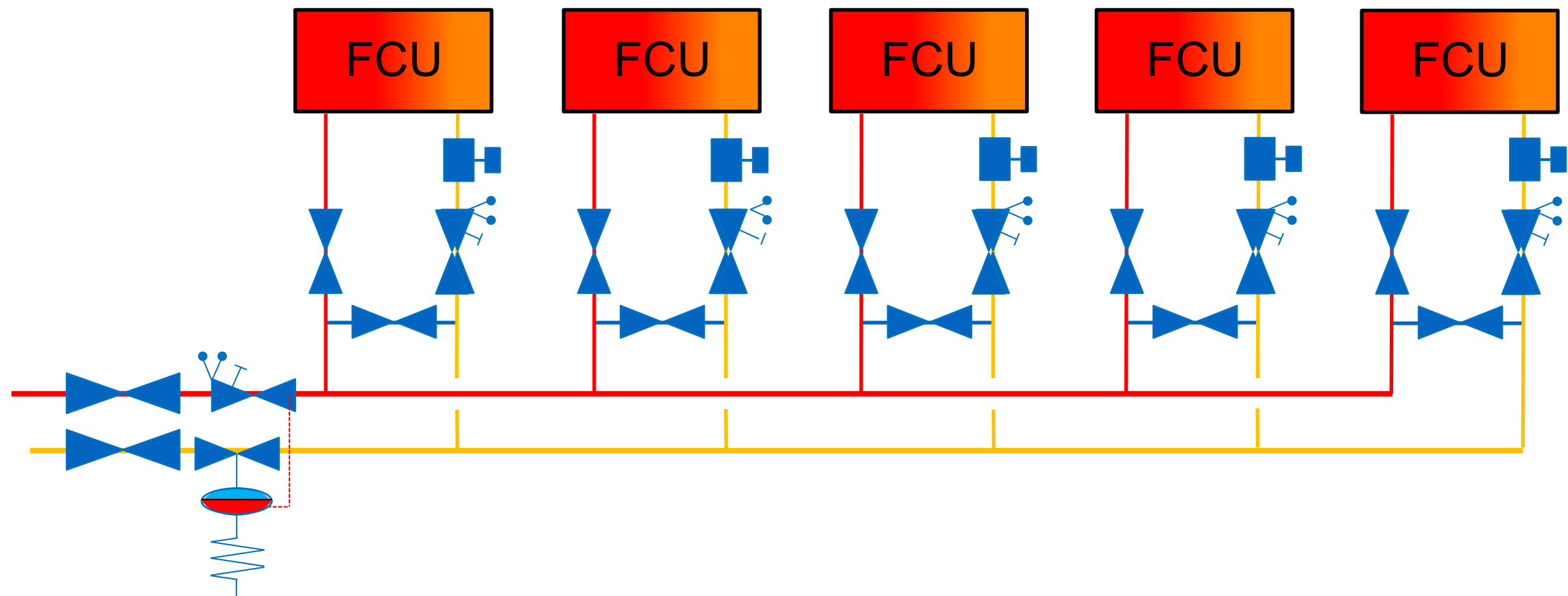
**Variable Flow Areas**

- chilled beams
- radiant panels
- FCUs
- AHUs

### Controlling Fluctuating System Pressures

Following the move to *variable volume* system design

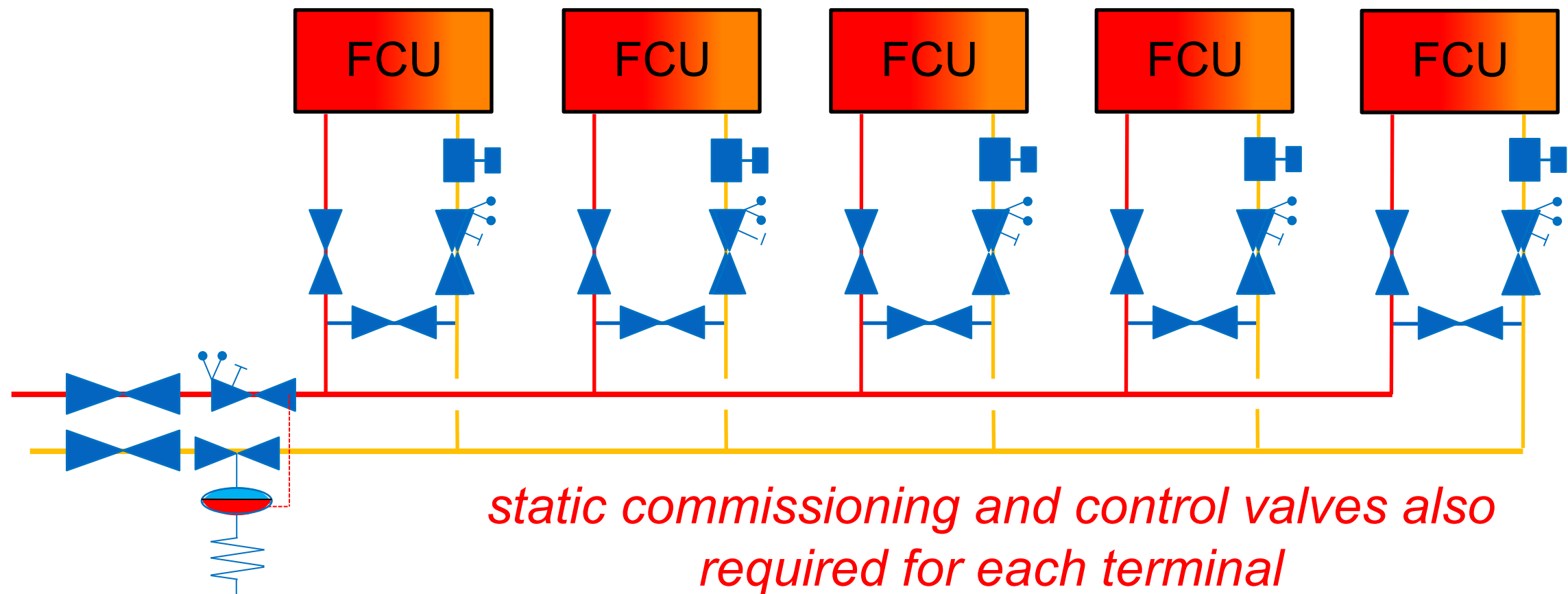
- DPCVs were used to create areas with stable pressures



## Controlling Fluctuating System Pressures

## DPCVs prove difficult to select correctly

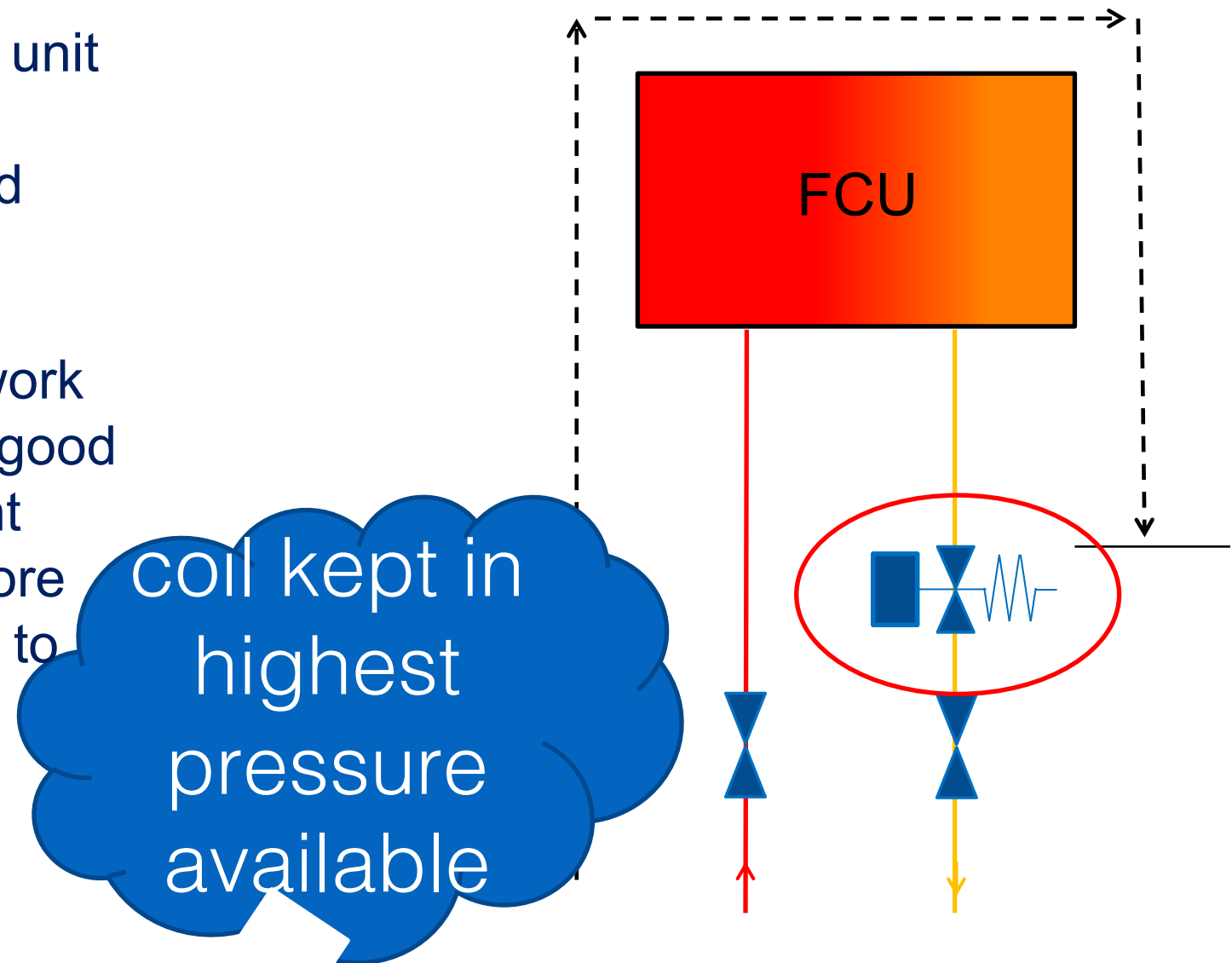
- sizing relatively easy
- gathering required information difficult / impossible
  - number of terminals in protected zone / controlled  $D_p$  ?





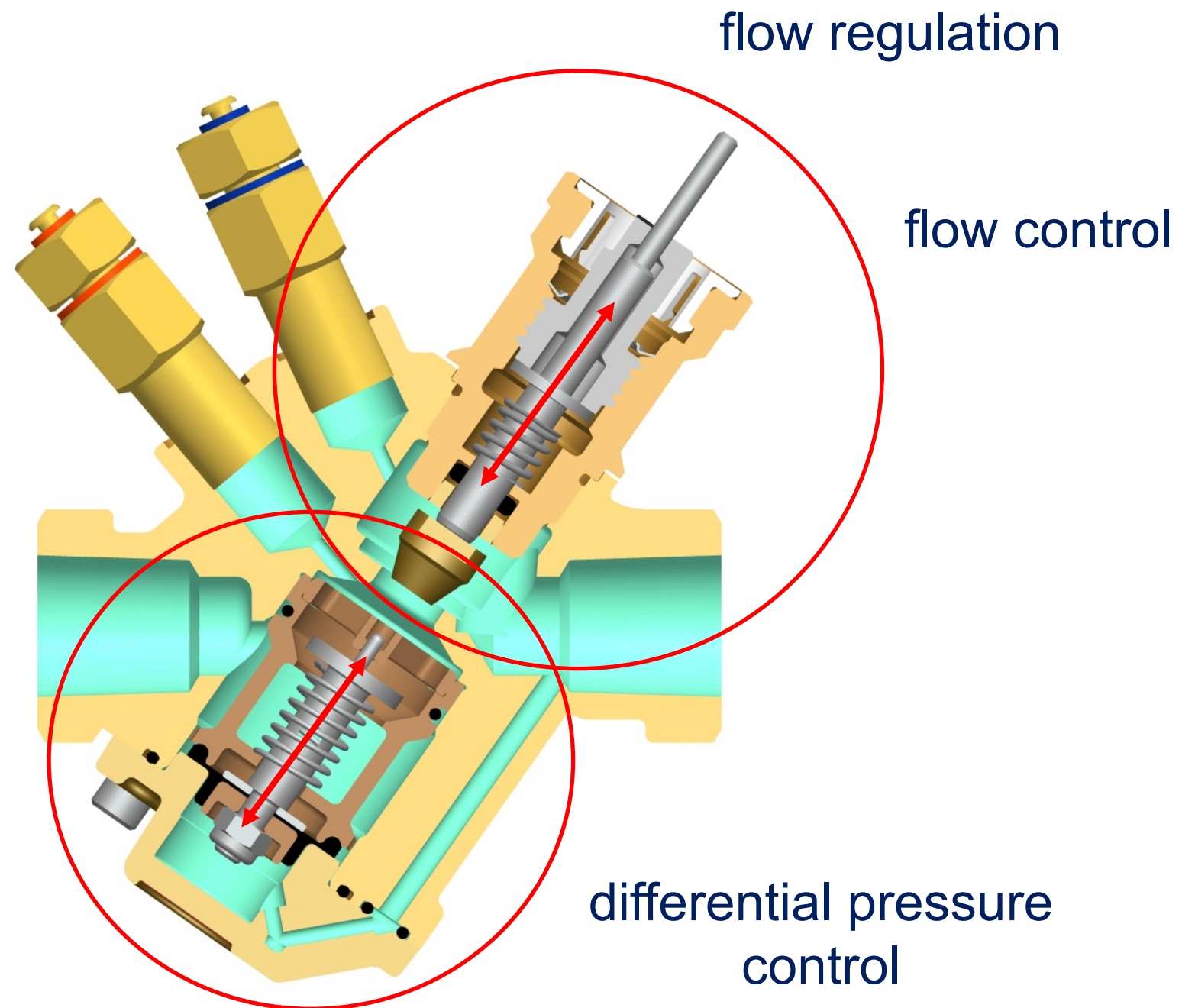
## Controlling Fluctuating System Pressures

- Normally fitted to each terminal unit
- Return pipework is the preferred position
- Clear / straight lengths of pipework are *not* required for PICVs but 'good practice' suggests some straight lengths of pipework create a more stable flow pattern for the PICV to control



What is a PICV?

*Three functions in one*



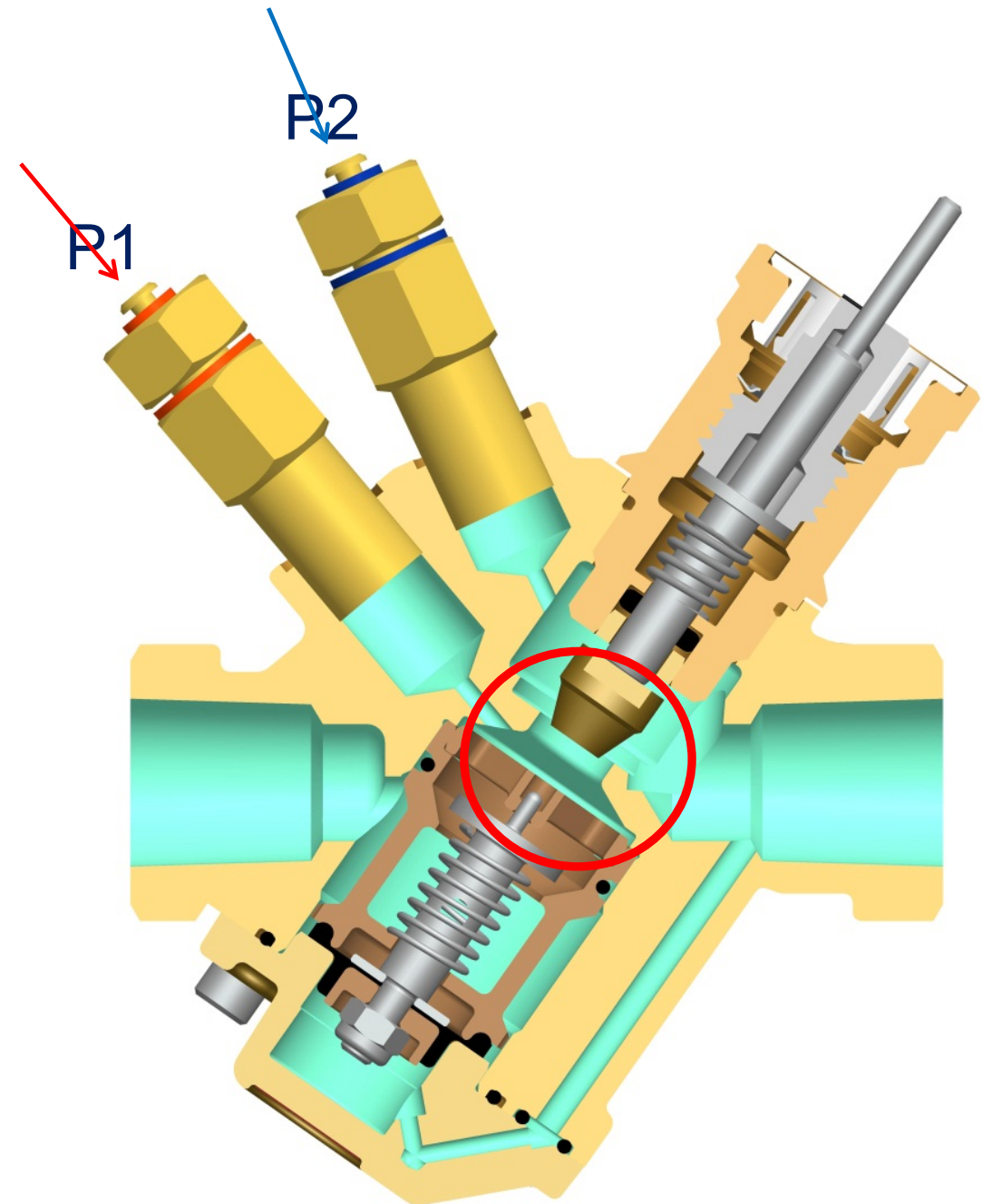


## Pressure Drops Across PICV

Differential pressure controller holds the Dp across the seat constant

Dp confirmed by test points

*Often referred to as P1 – P2*

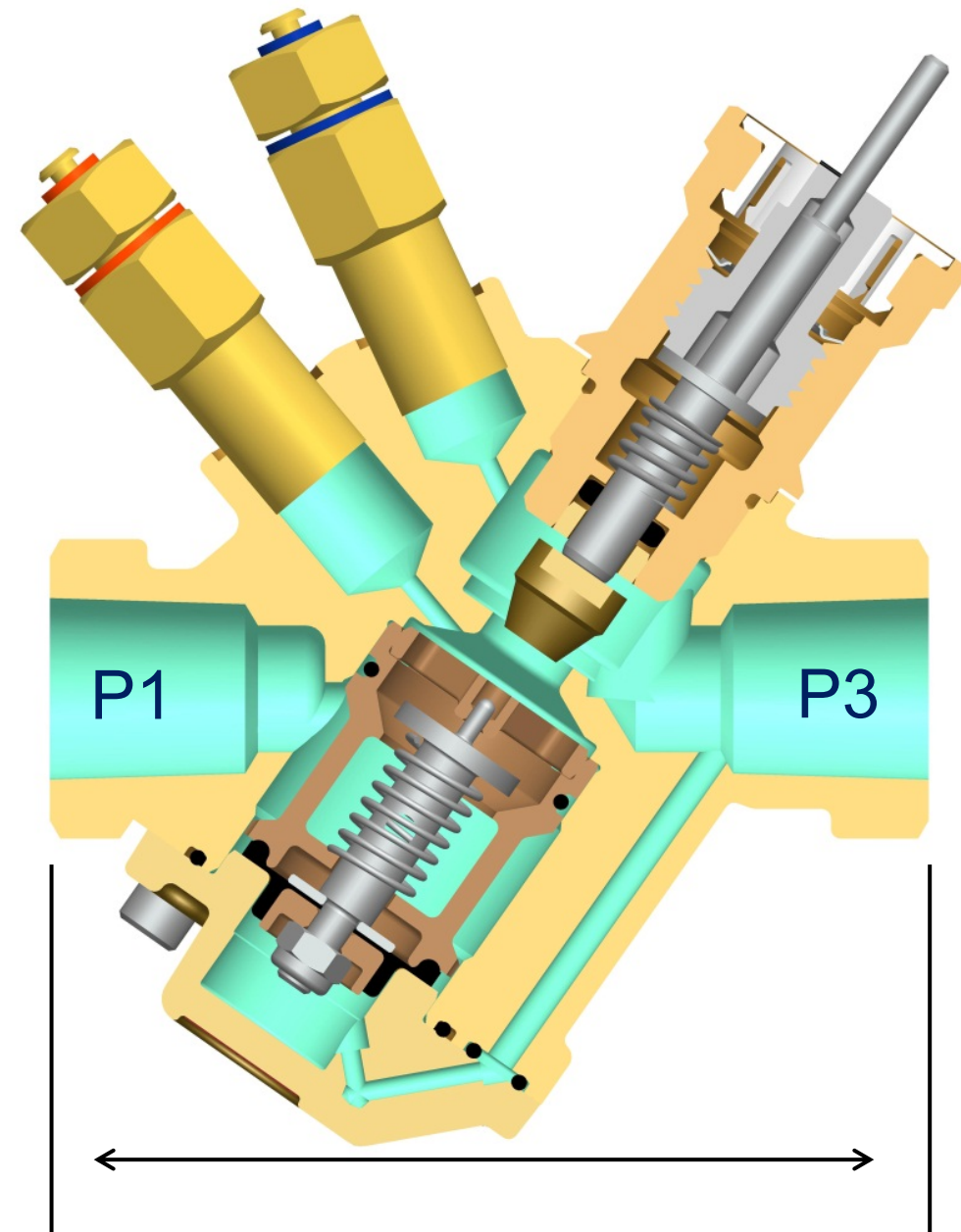


## Pressure Drops Across PICV

Differential pressure  
across the PICV  
varies

On smaller sizes  
total Dp  
**NOT**  
confirmed by test  
points

*Often referred to  
as P1 – P3*

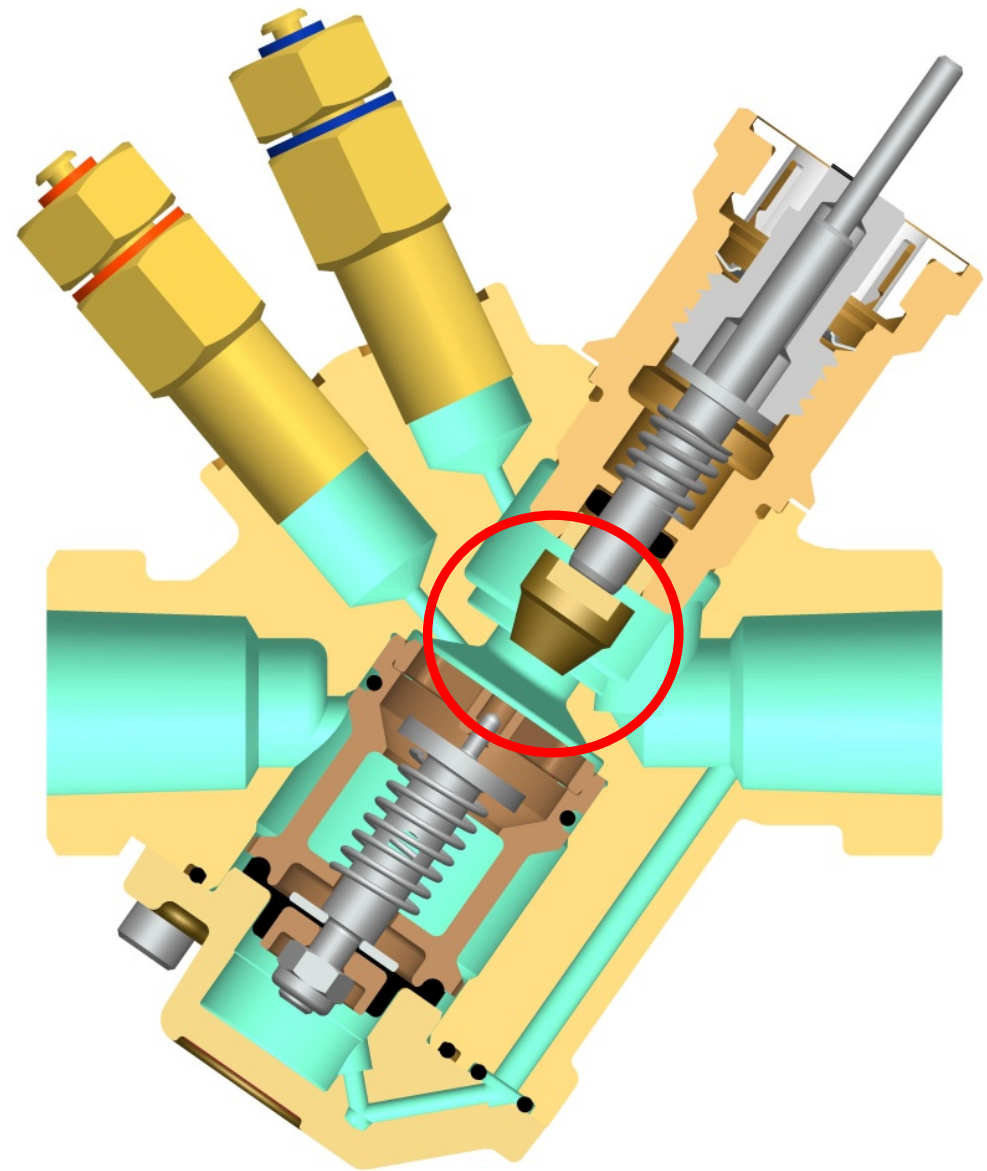




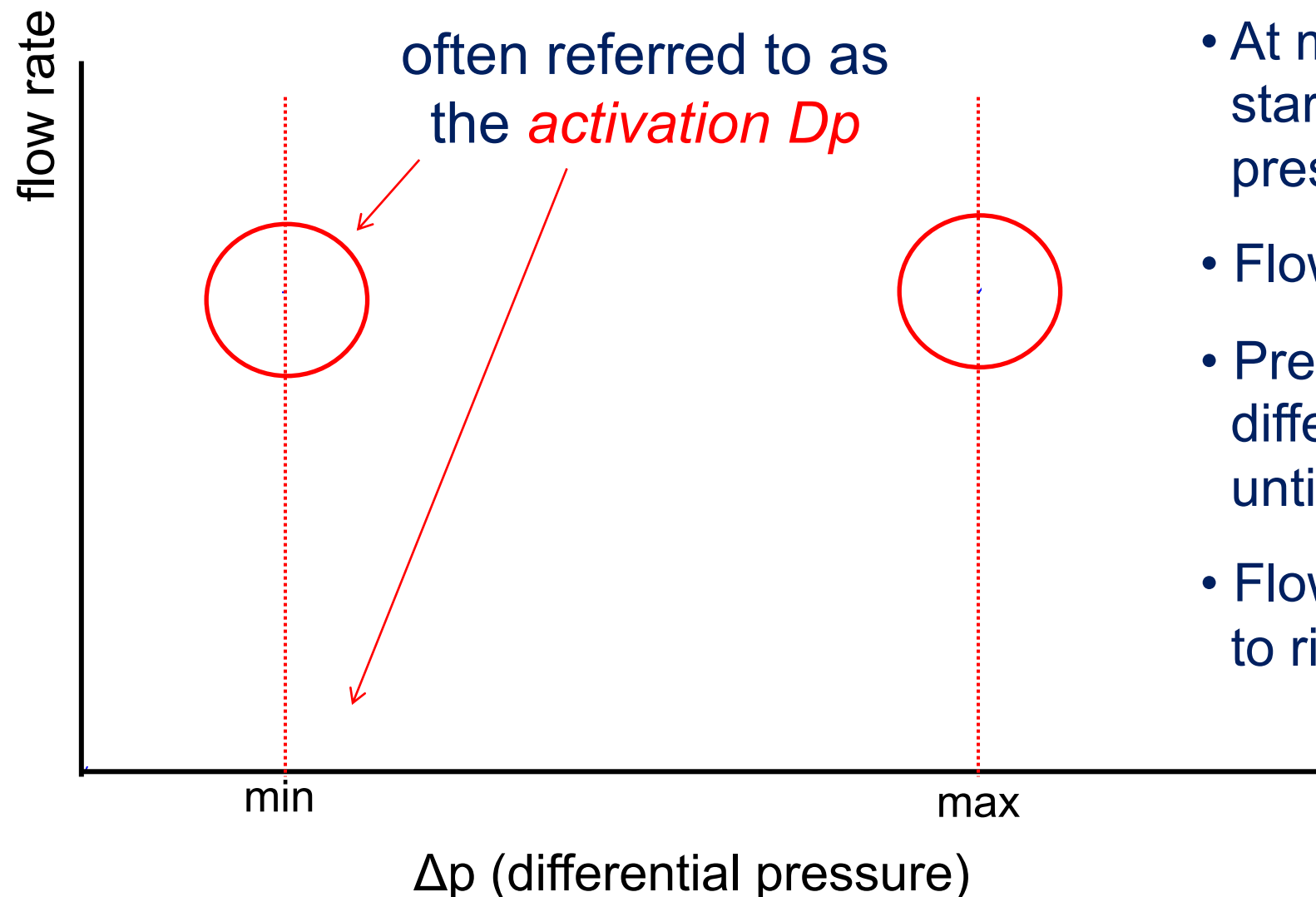
$$\text{Authority } h = \frac{\text{Dp across control valve}}{\text{DP across circuit}}$$

$$\text{Dp control valve} = \text{DP circuit}$$

*Considered to be '1'*



### PICV Activation



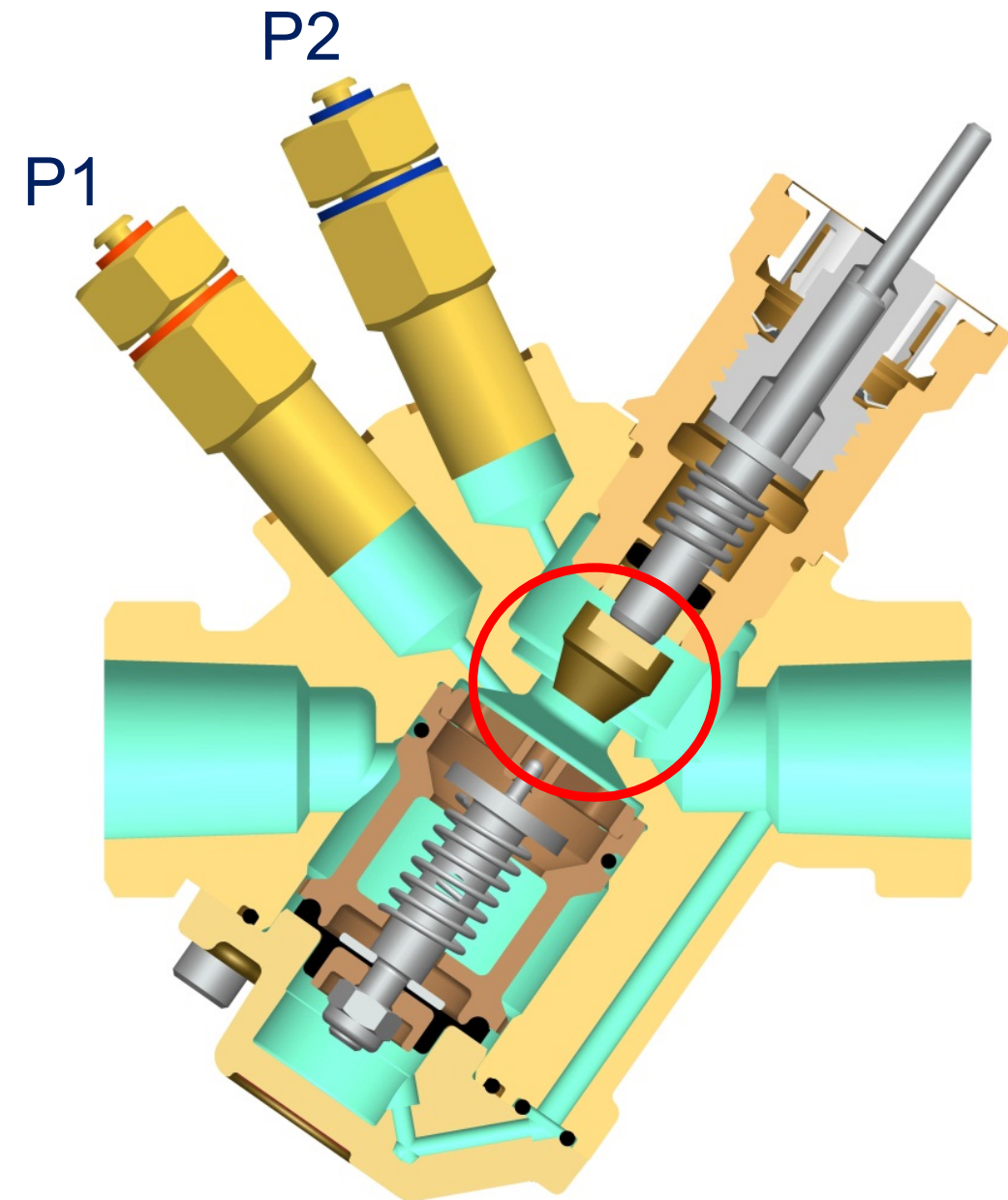
- As flow increases  $\Delta p$  increases
- At min  $\Delta p$ , pressure controller starts to hold differential seat pressure constant
- Flow rate remains constant
- Pressure controller controls seat differential pressure at *min  $\Delta p$*  until *max total  $\Delta p$*  is reached
- Flow rate will rise as  $\Delta p$  continues to rise

## PICV Activation

P1 to P2

*the activation  $\Delta p$   
always constant  
measured by integral test points*

*Where test points measure  
P1 – P3  
 $\Delta p$  will vary  
between min & max*

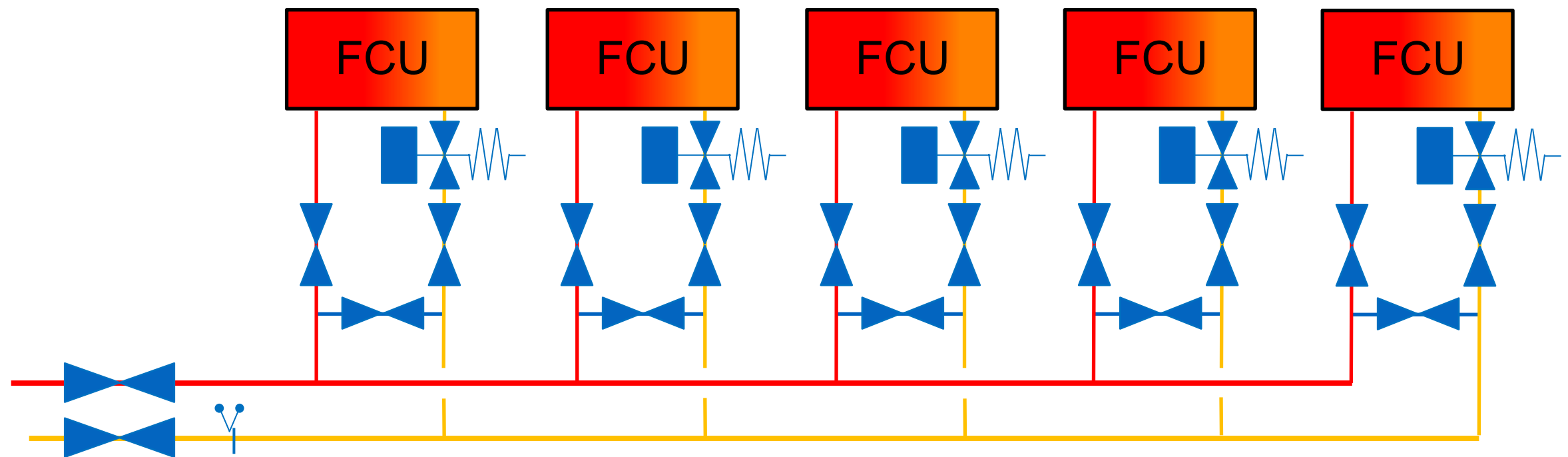


### PICV Selection

Flow rate is the *main* selection criteria

Other considerations could be

- minimising pressure drop
- matching line sizes



*Almost always it is flow rate that determines selection*

### PICV Selection

DN15 Low Flow			DN15 Standard Flow		
setting position	flow rate l/s	total $\Delta p$ kPa	setting position	flow rate l/s	total $\Delta p$ kPa
2	0.008	15	2	0.04	15
3	0.010	15	3	0.060	15
4	0.020	15	4	0.080	20
5	0.030	15	5	0.105	20
6	0.040	20	6	0.120	20
7	0.050	20	7	0.140	20
8	0.060	20	8	0.155	25
9	0.070	20	9	0.175	25
10	0.080	20	10	0.200	25

selection charts are available

- flow rates
- set position
- minimum  $\Delta p$  (differential pressure) *total loss at activation*

### PICV Selection – activation point

- *activation*  $\Delta p$  lower than *total*  $\Delta p$
- activation  $\Delta p$  constant
- total  $\Delta p$  varies

DN15 standard Flow			
setting position	flow rate l/s	total $\Delta p$ kPa	activation $\Delta p$ kPa
2	0.040	15	12
3	0.060	15	12
4	0.080	20	12
5	0.100	20	12
6	0.120	20	12
7	0.140	20	12
8	0.160	25	12
9	0.180	25	12
10	0.200	25	12



### PICV Selection

DN15 Low Flow			DN15 Standard Flow			DN20 Standard Flow		
setting position	flow rate l/s	total $\Delta p$ kPa	setting position	flow rate l/s	total $\Delta p$ kPa	setting position	flow rate l/s	total $\Delta p$ kPa
2	0.008	15	2	0.04	15	2	0.065	15
3	0.010	15	3	0.060	15	3	0.100	20
4	0.020	15	4	0.080	20	4	0.130	20
5	0.030	15	5	0.105	20	5	0.160	20
6	0.040	20	6	0.120	20	6	0.190	25
7	0.050	20	7	0.140	20	7	0.220	25
8	0.060	20	8	0.155	25	8	0.240	25
9	0.070	20	9	0.175	25	9	0.260	25
10	0.080	20	10	0.200	25	10	0.280	25

Example  
required flow rate 0.160l/s

between setting 8 & 9  
8.3ish

setting 5

### PICV Selection

DN15 Low Flow			DN15 Standard Flow			DN20 Standard Flow		
setting position	flow rate l/s	total $\Delta p$ kPa	setting position	flow rate l/s	total $\Delta p$ kPa	setting position	flow rate l/s	total $\Delta p$ kPa
2	0.008	15	2	0.04	15	2	0.065	15
3	0.010	15	3	0.060	15	3	0.100	20
4	0.020	15	4	0.080	20	4	0.130	20
5	0.030	15	5	0.105	20	5	0.160	20
6	0.040	20	6	0.120	20	6	0.190	25
7	0.050	20	7	0.140	20	7	0.220	25
8	0.060	20	8	0.155	25	8	0.240	25
9	0.070	20	9	0.175	25	9	0.260	25
10	0.080	20	10	0.200	25	10	0.280	25

example  
required flow rate 0.160l/s

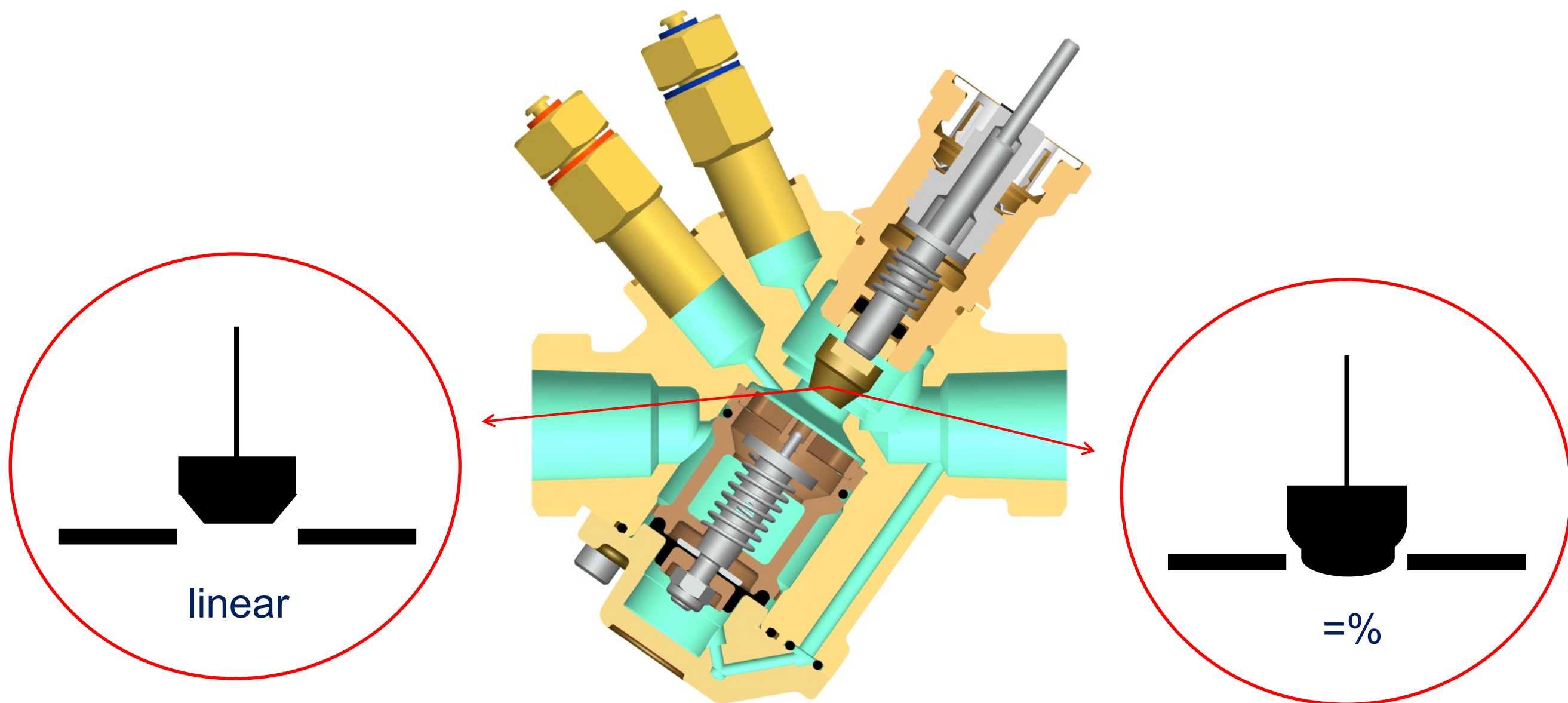
total  $\Delta p$  (P1 – P3)  
25 kPa

total  $\Delta p$  (P1 – P3)  
20 kPa

*Total  $\Delta p$  could rise to max  $\Delta p$  – 350 kPa*

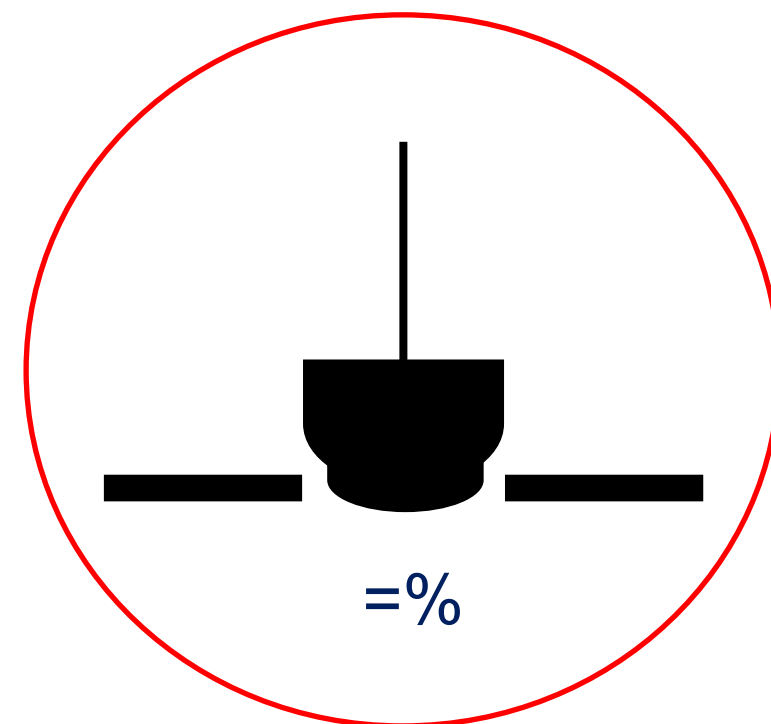
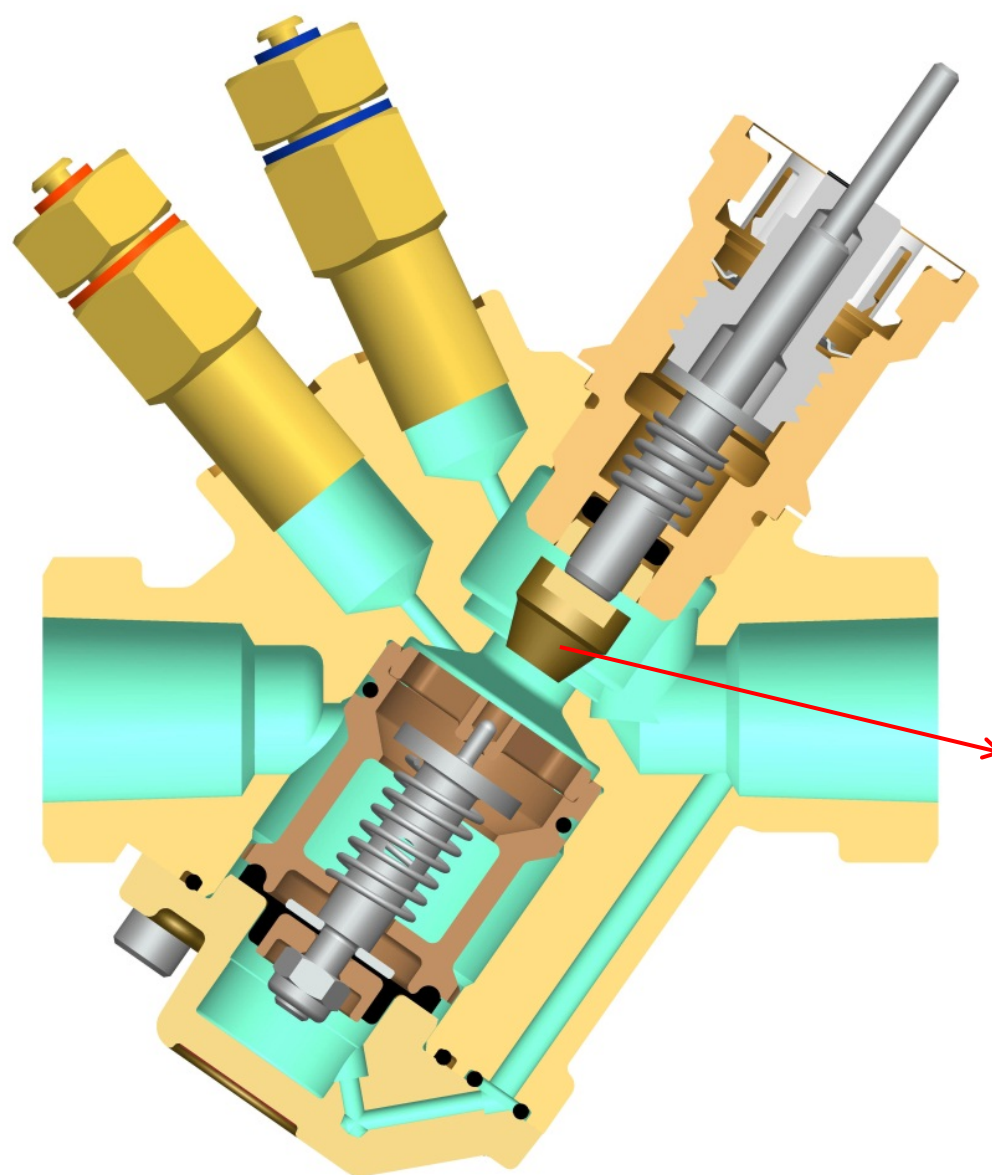


## PICV Control Characteristic

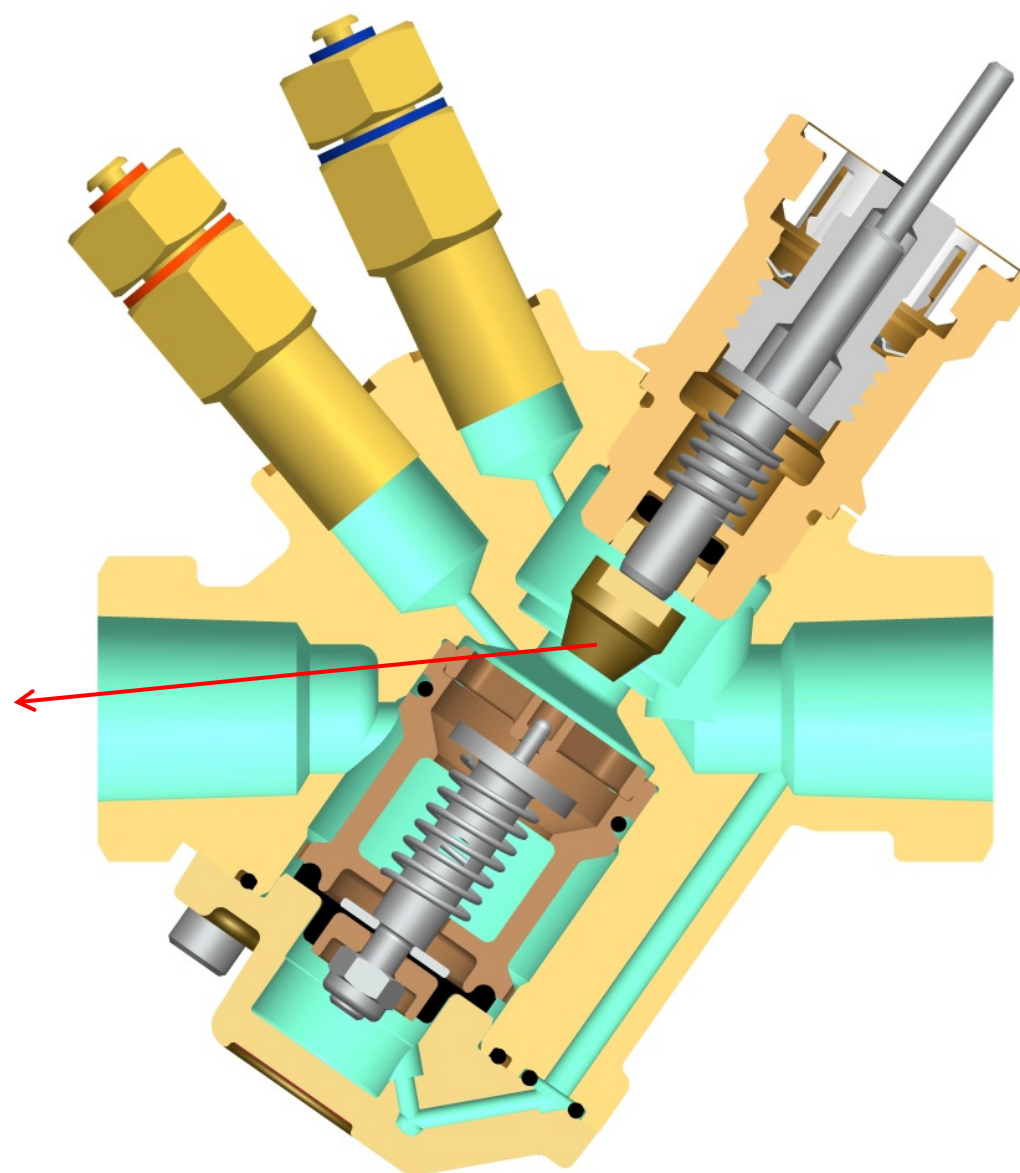
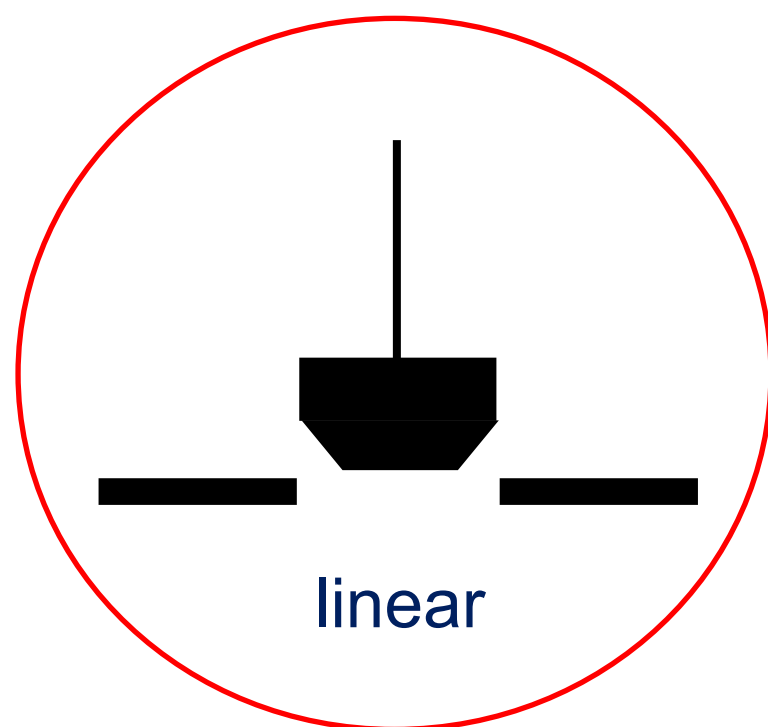


## PICV Control Characteristic - =%

if stem it used to  
regulate flow rate  
**Control Characteristic**  
is changed at lower  
settings



PICV Control Characteristic - linear



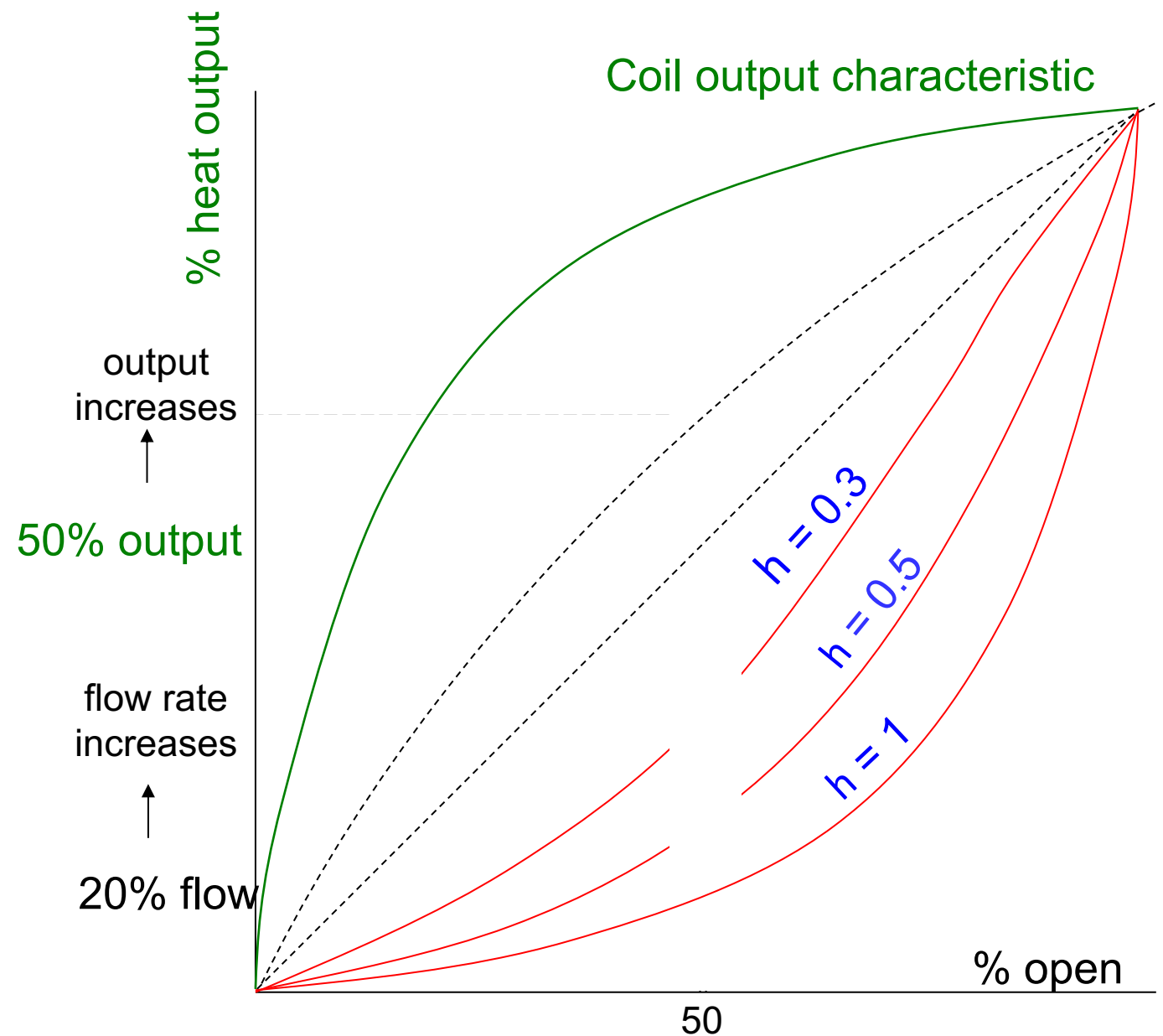
If stem it used to  
regulate flow rate  
**Control Characteristic**  
is **NOT** changed at  
lower settings

## PICV Control Characteristic

Equal percentage control valves are designed with a

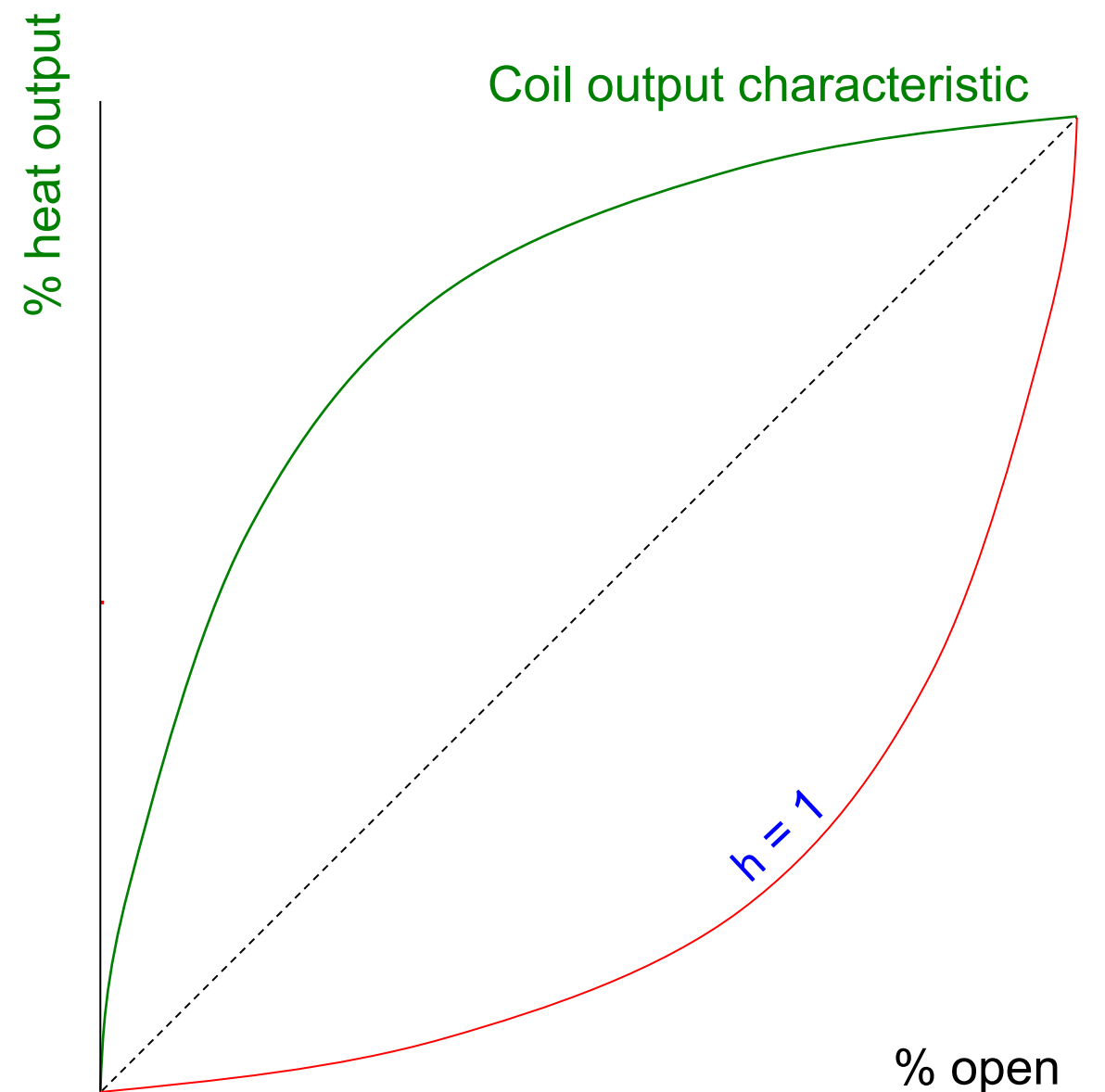
$$h = 1$$

Mirror image of coil characteristic



## PICV Control Characteristic - =%

- Establish a direct relationship between open position & *heat output*
- 50% valve opening = 50% heat output



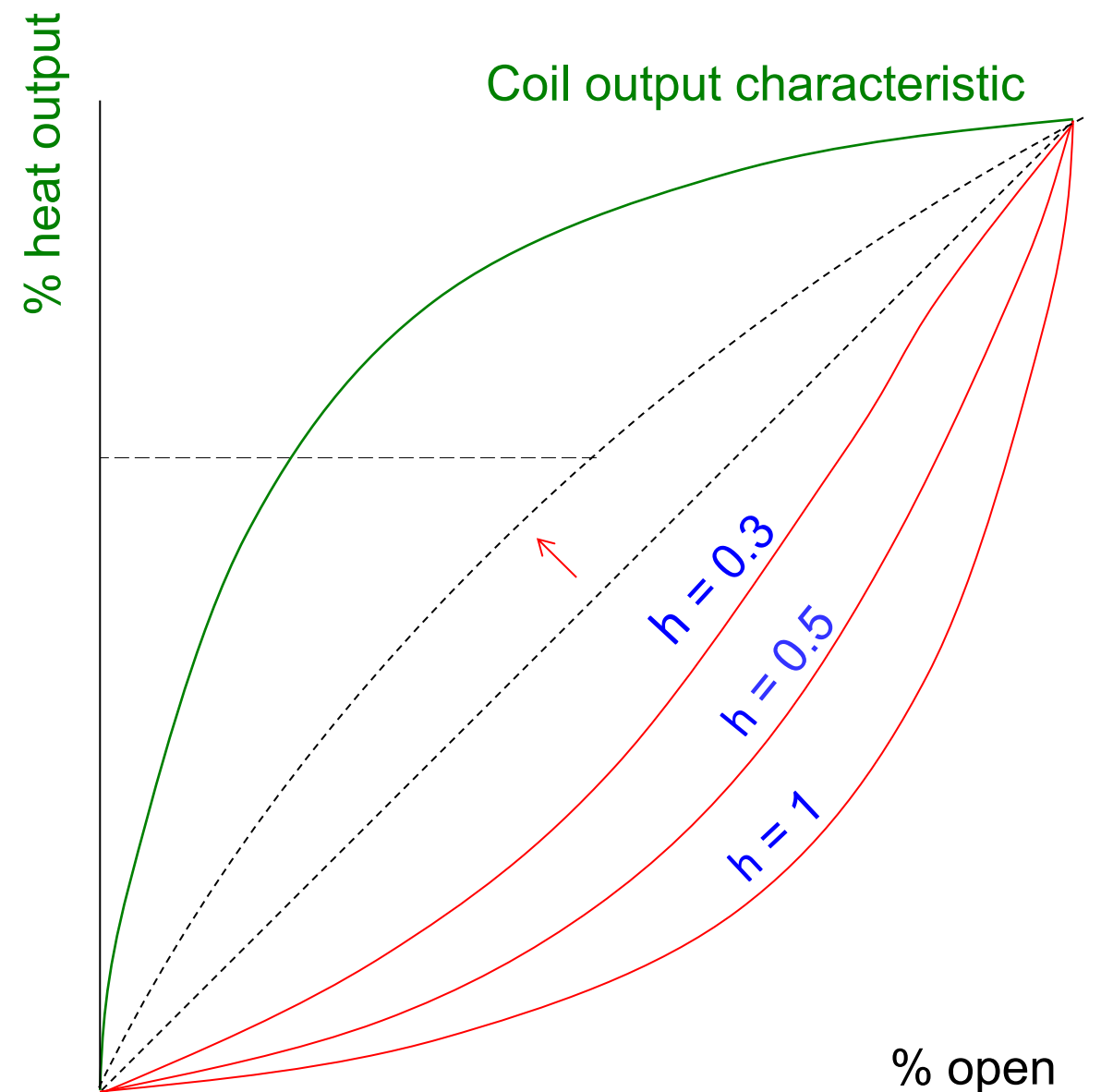


## PICV Control Characteristic - %

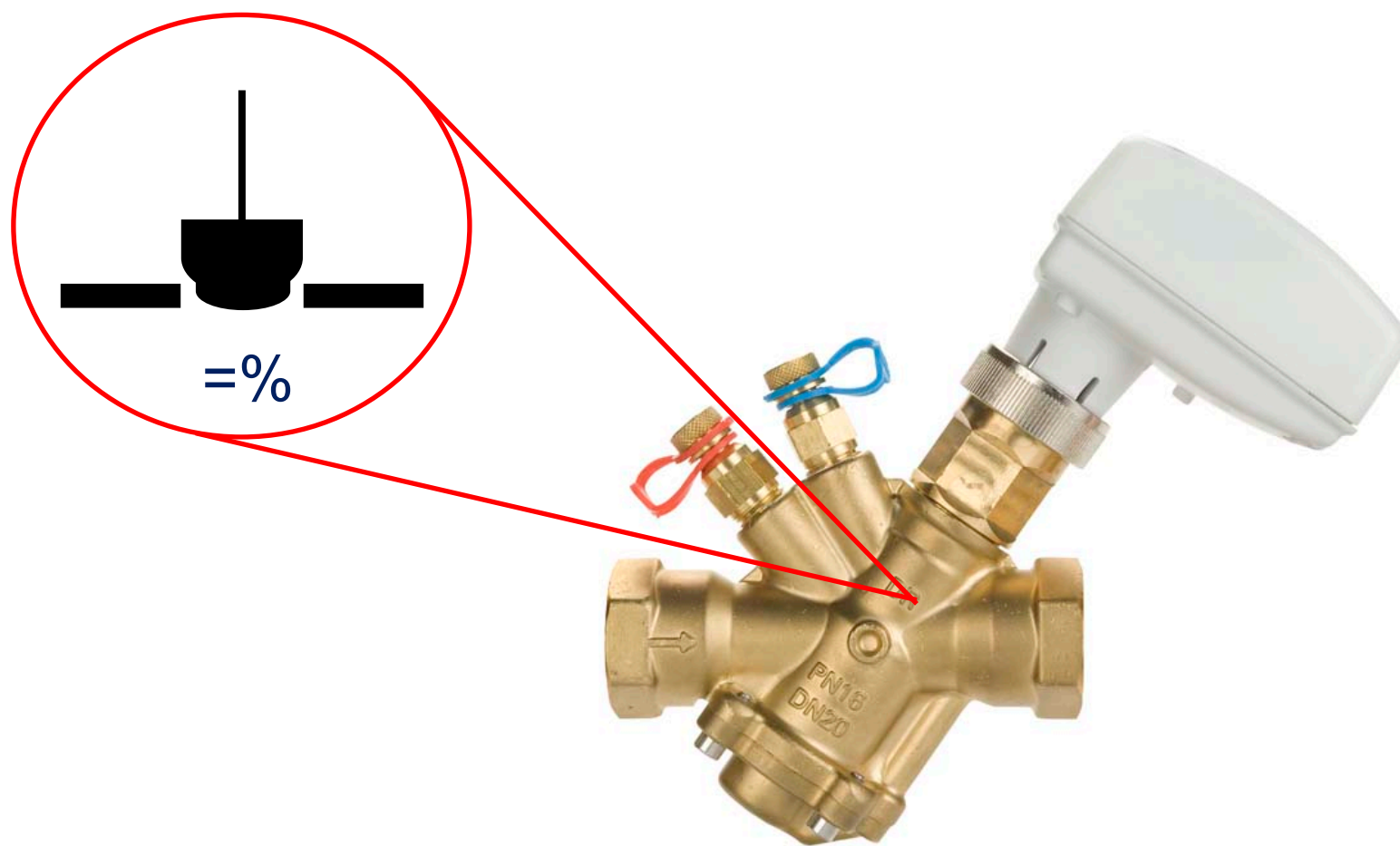
## Dynamic Balancing

- Where stem used to regulate flow rate
- Changes control characteristic

*combined 'linear' position moves  
due to effect of valve  
characteristic*

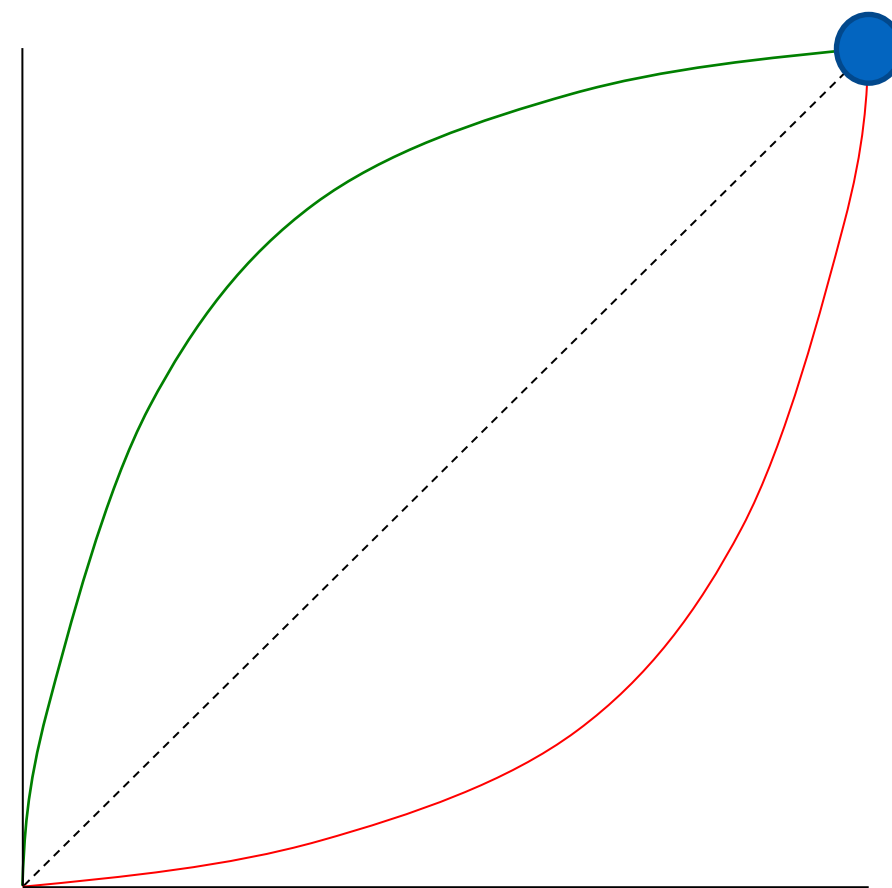


## PICV Control Characteristic - =%



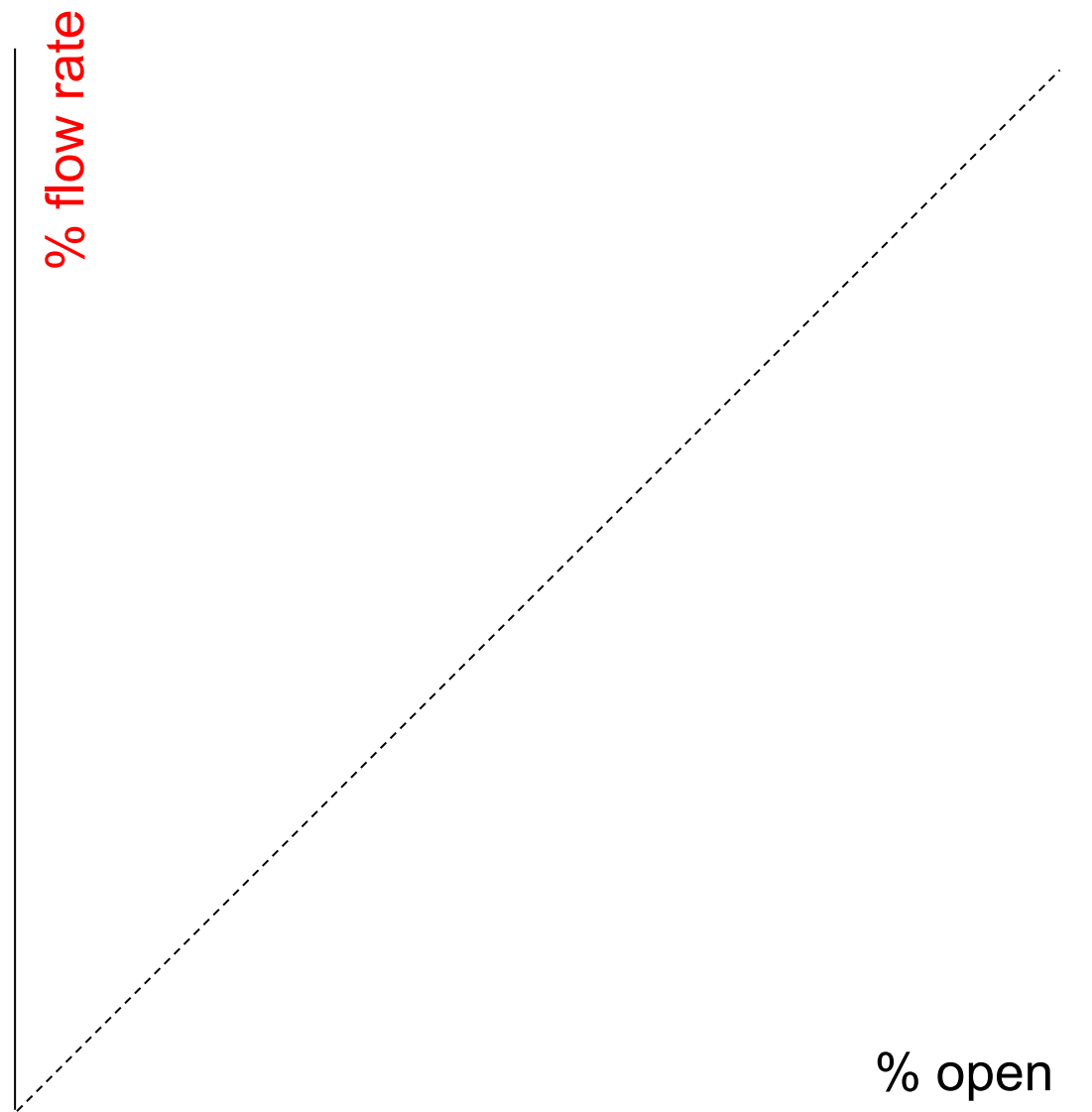
*'Linear' control actuator giving equal percentage characteristic*

## Dynamic Balancing



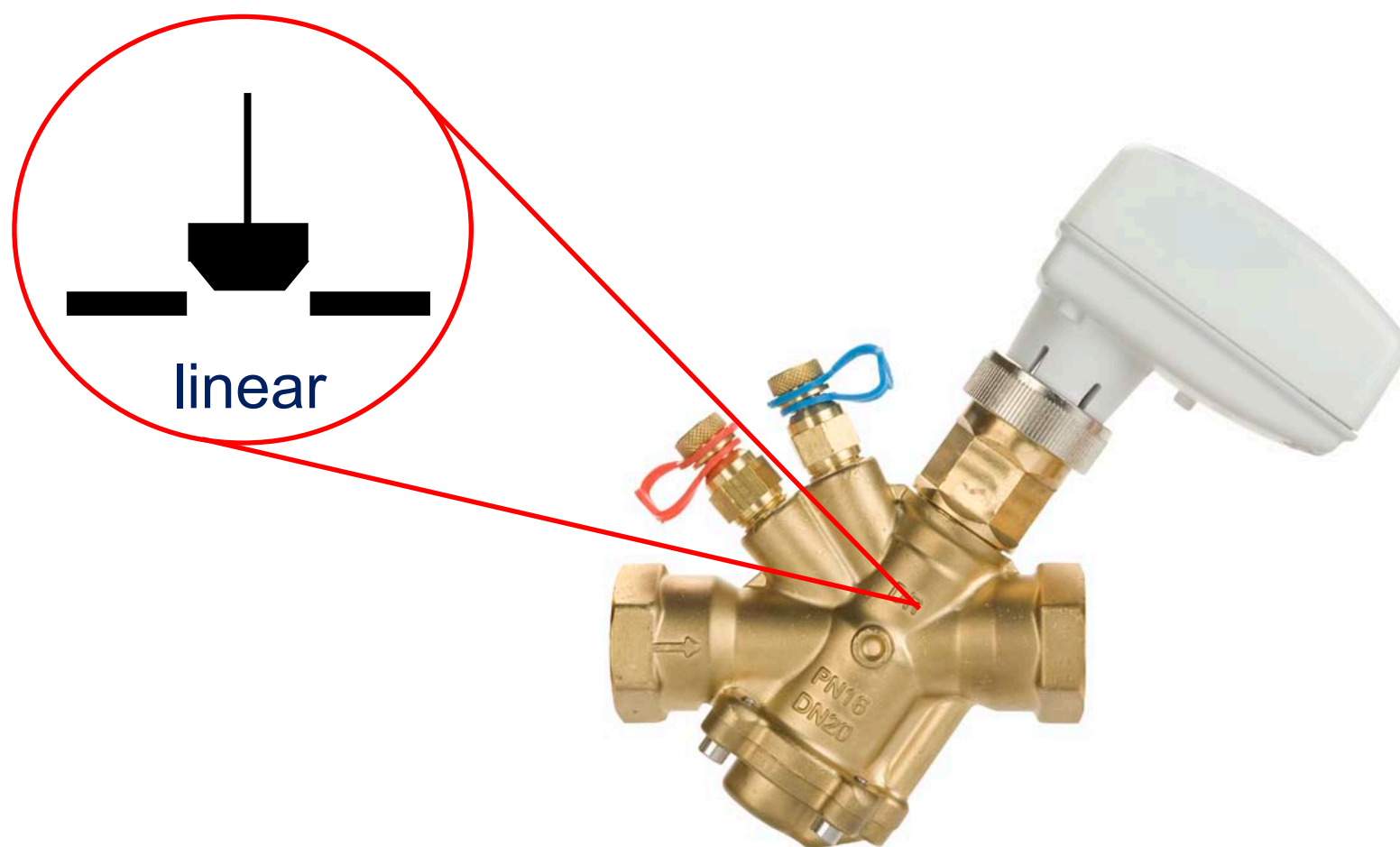
## PICV Control Characteristic - linear

- Establish a direct relationship between open position & *flow rate*
- 50% valve opening = 50% *flow rate*
- Not effected by flow regulation

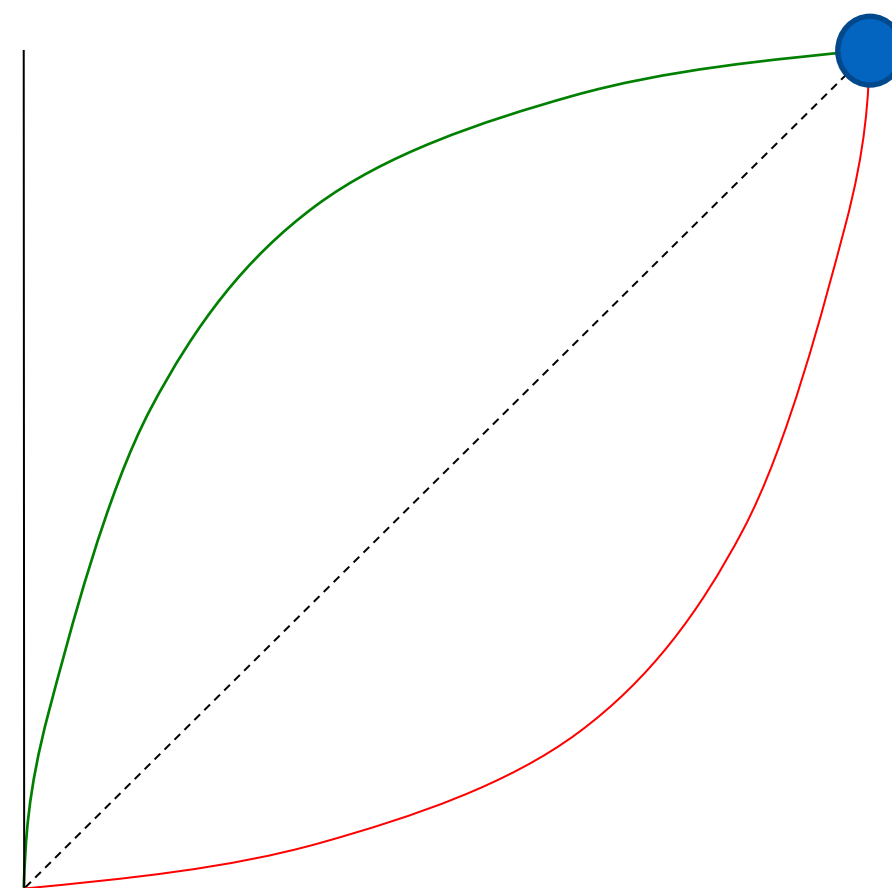




### PICV Control Characteristic - linear



*'Equal Percentage' control actuator giving equal percentage characteristic*



## Variable Volume System

At maximum pump turndown, typically 10 - 20%, consideration needs to be given to branches to ensure

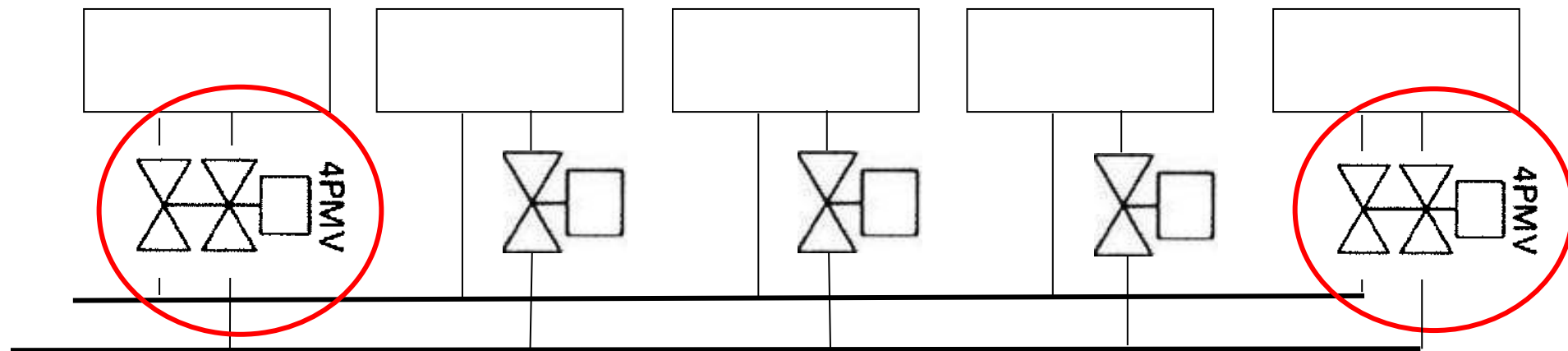
- pump flow at minimum load
- circulation of water treatment
- ready supply of heating / chilled water

## Dynamic Balancing of the System

### Possible solution

End terminal could have a 3 or 4 port control valve

- on larger circuits additional 3 or 4 ports could be added



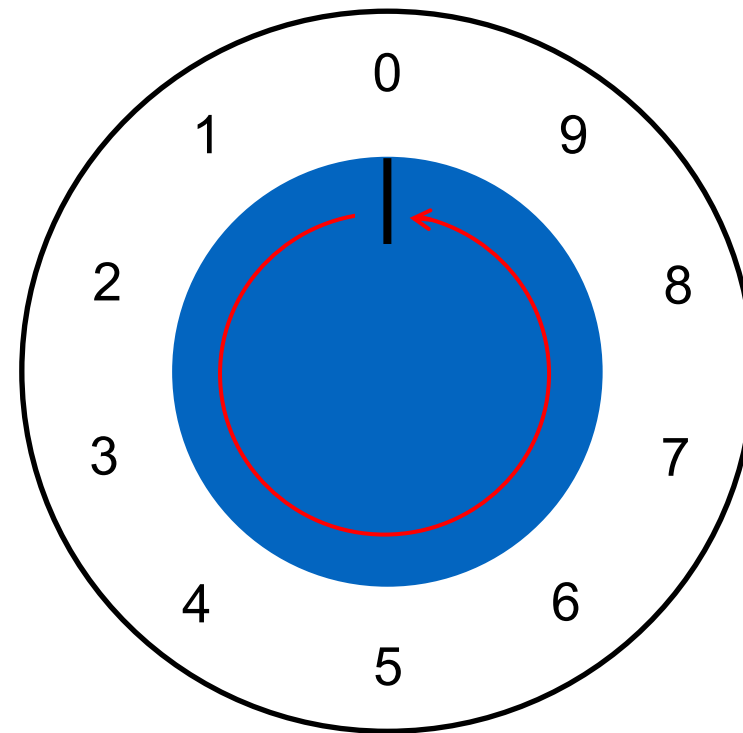
## Flow Accuracy

Different factors affect accuracy

- pre-set position
- available pump pressure
- actuator control

*Accuracy is a combination of all factors*

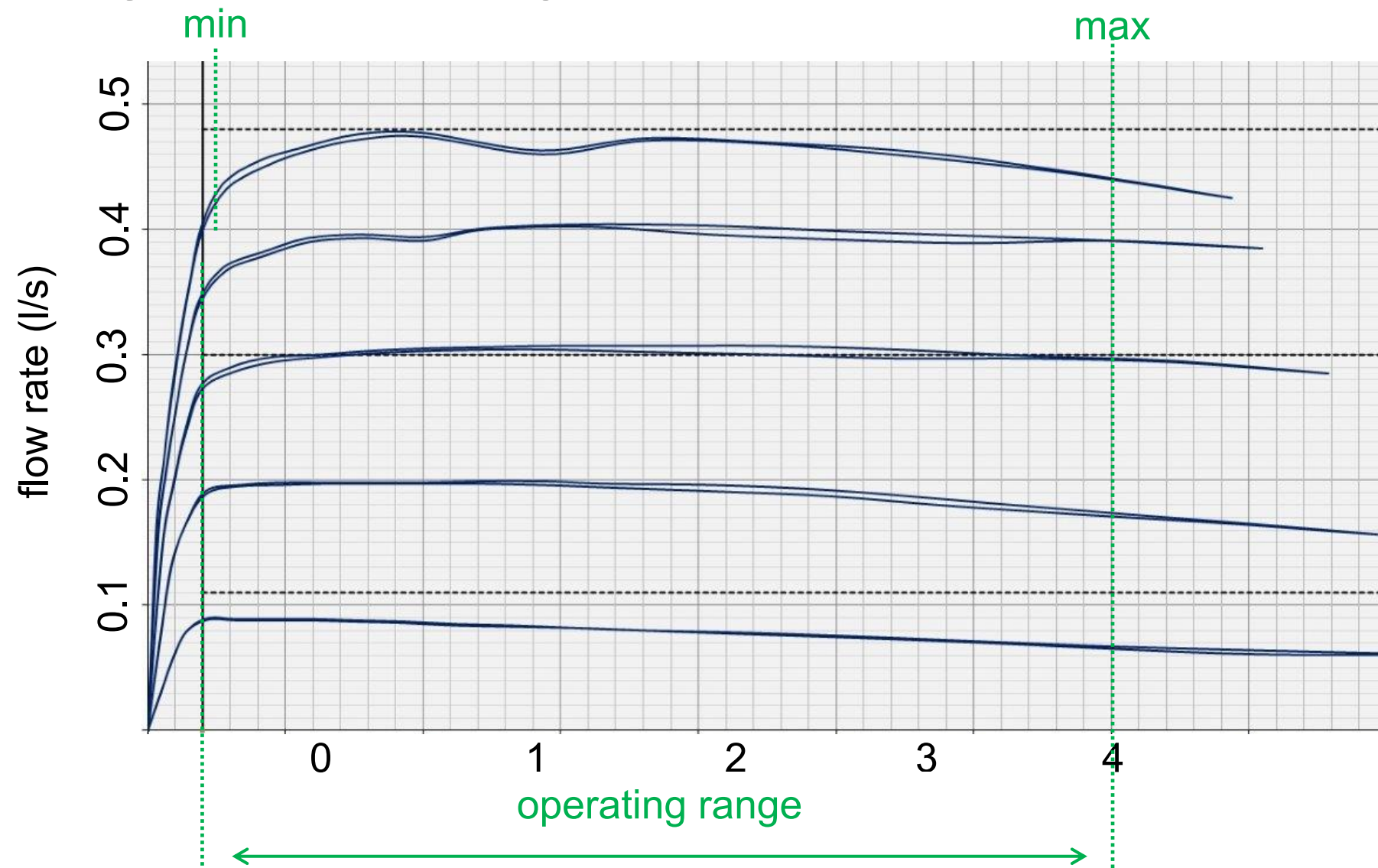
## Flow Accuracy – Pre-set Position



Aligning dial with scale

*Pre-set position accuracy can be improved by the installation of a FMD*

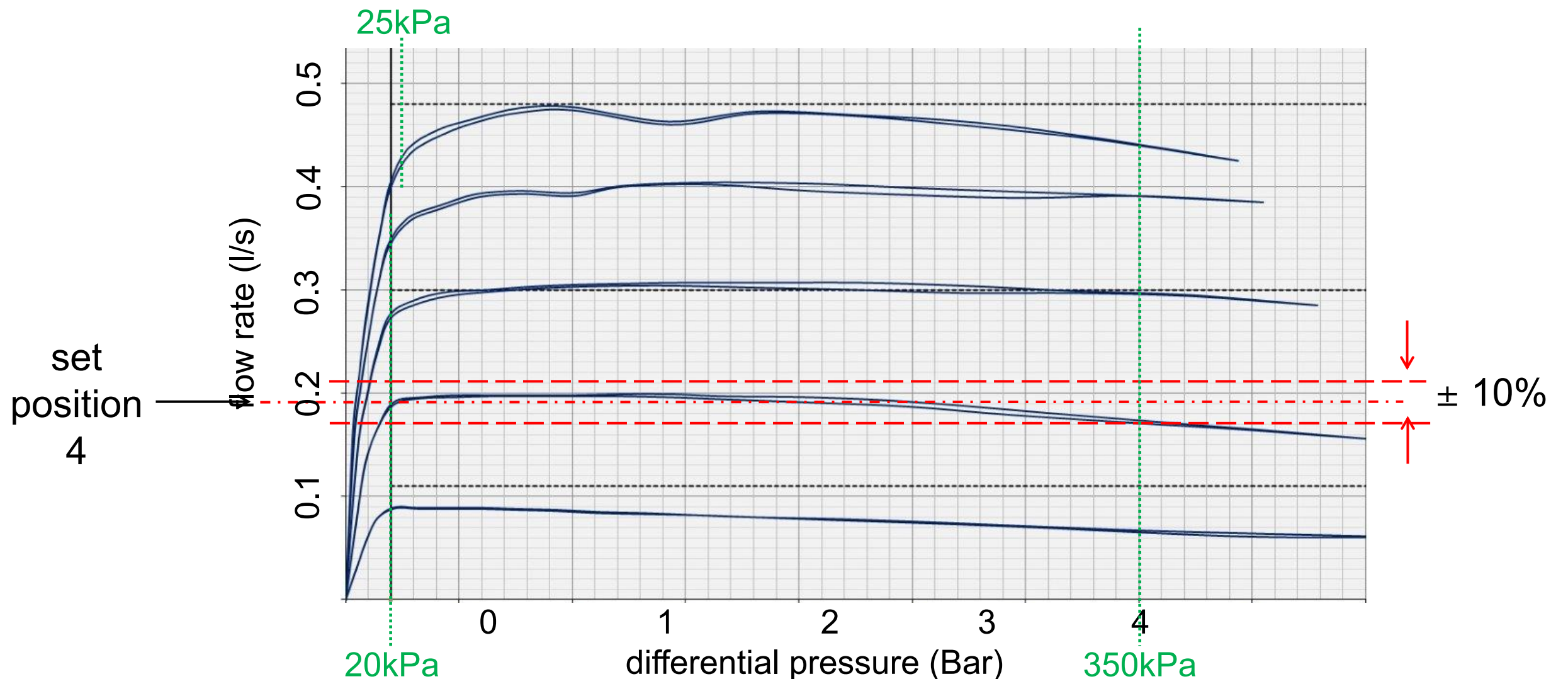
Flow Accuracy – Available Pump Pressure



*As pump pressure varies flow rate varies*



## Flow Accuracy – Available Pump Pressure



*As pump pressure varies flow rate varies*

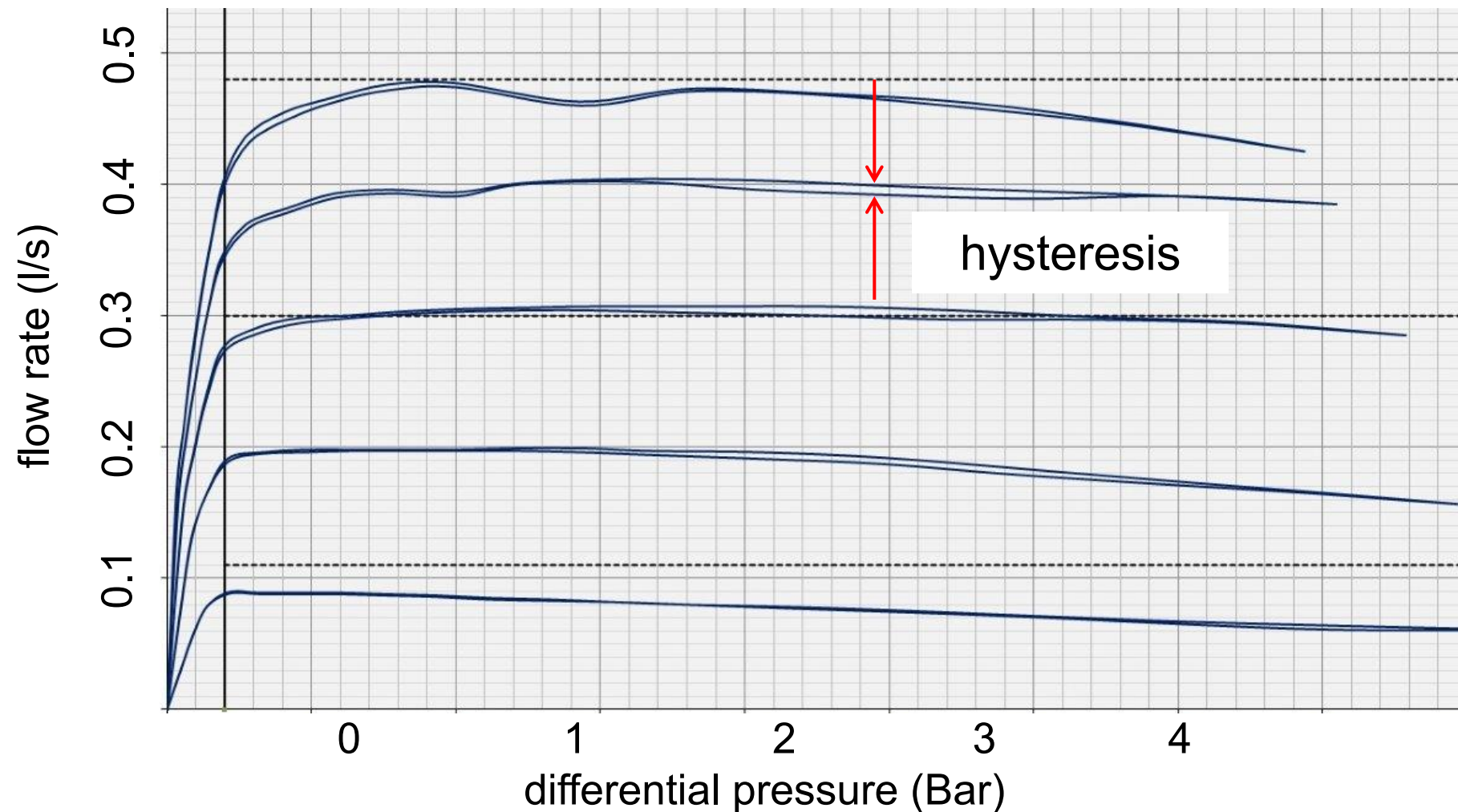


Top Solution, Top Choice

# TOP FLOW

## Dynamic Balancing

### Flow Accuracy – Rising / Falling Pressure



*Hysteresis is relatively low but need to be considered and understood*

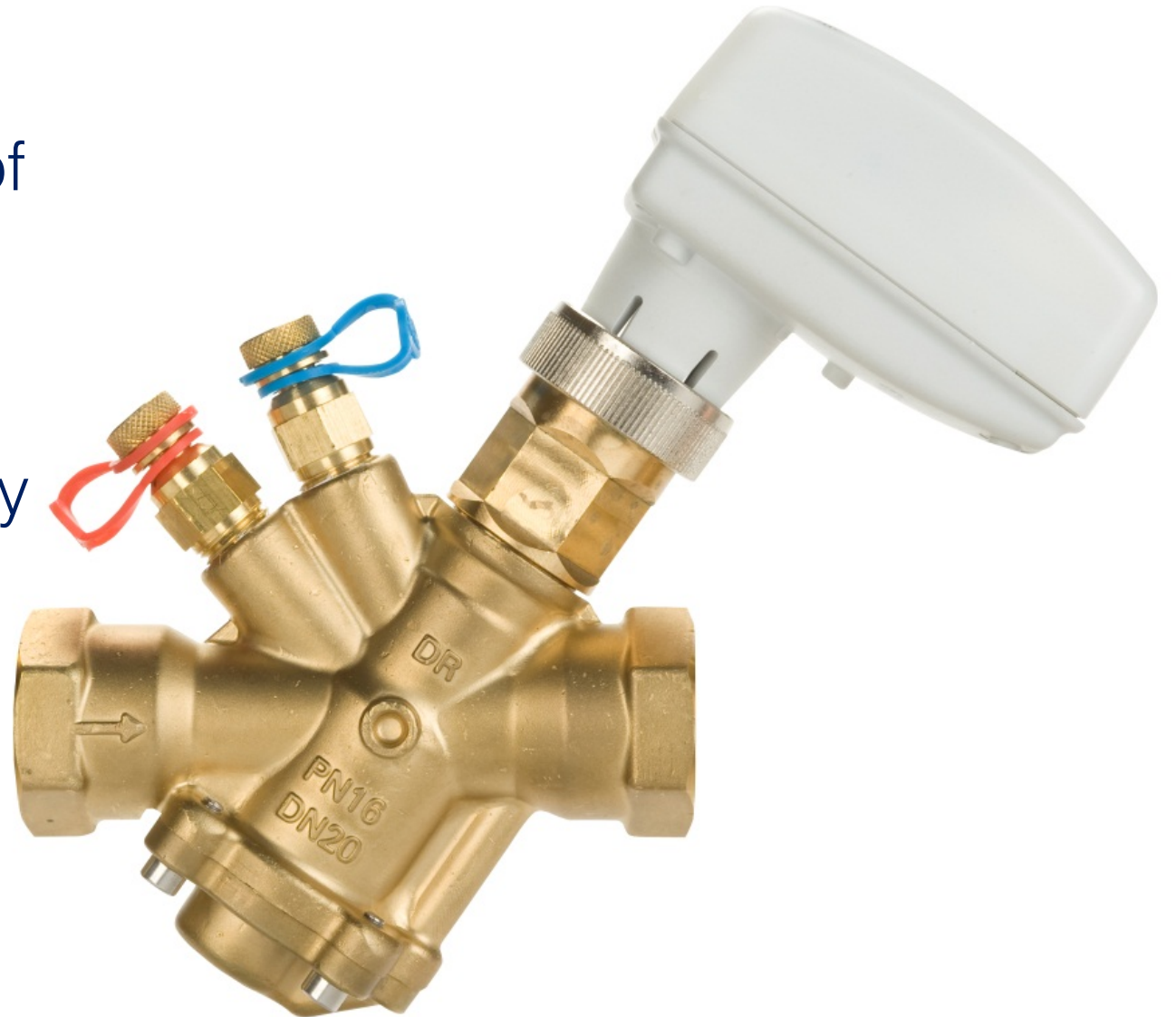


## Flow Accuracy – Actuator

Installed PICVs are a combination of

- PICV
- Actuator

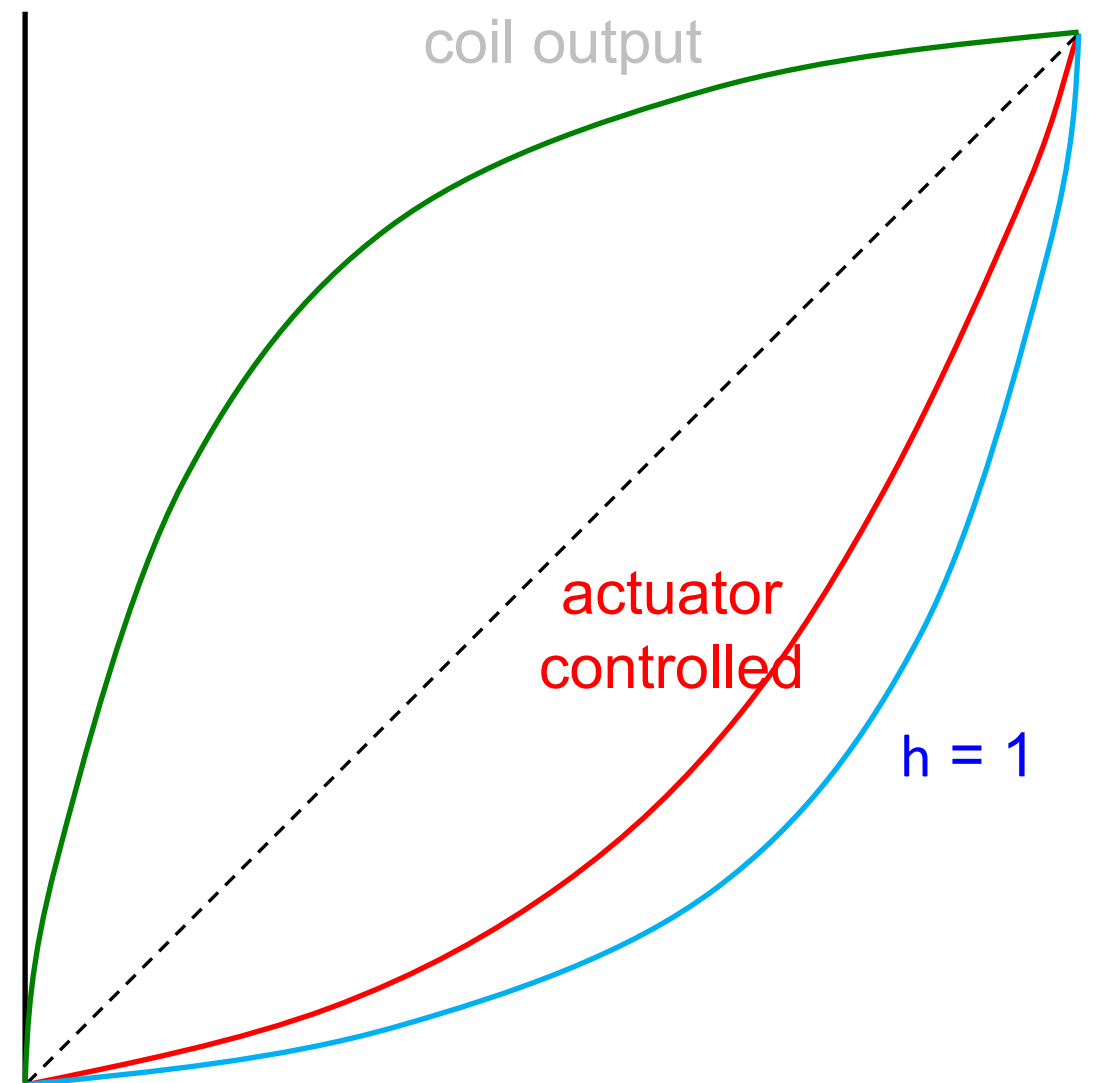
so we should consider the assembly  
and not the standalone PICV



*Consider as a 'matched' pair*

## Flow Accuracy – Actuator

- PICV can perform well as a stand alone valve
- But when actuator is fitted performance of valve can be undermined



## Actuators

Actuators are divided into 2 types

- Thermal
- Electro-mechanical

And then by control characteristic

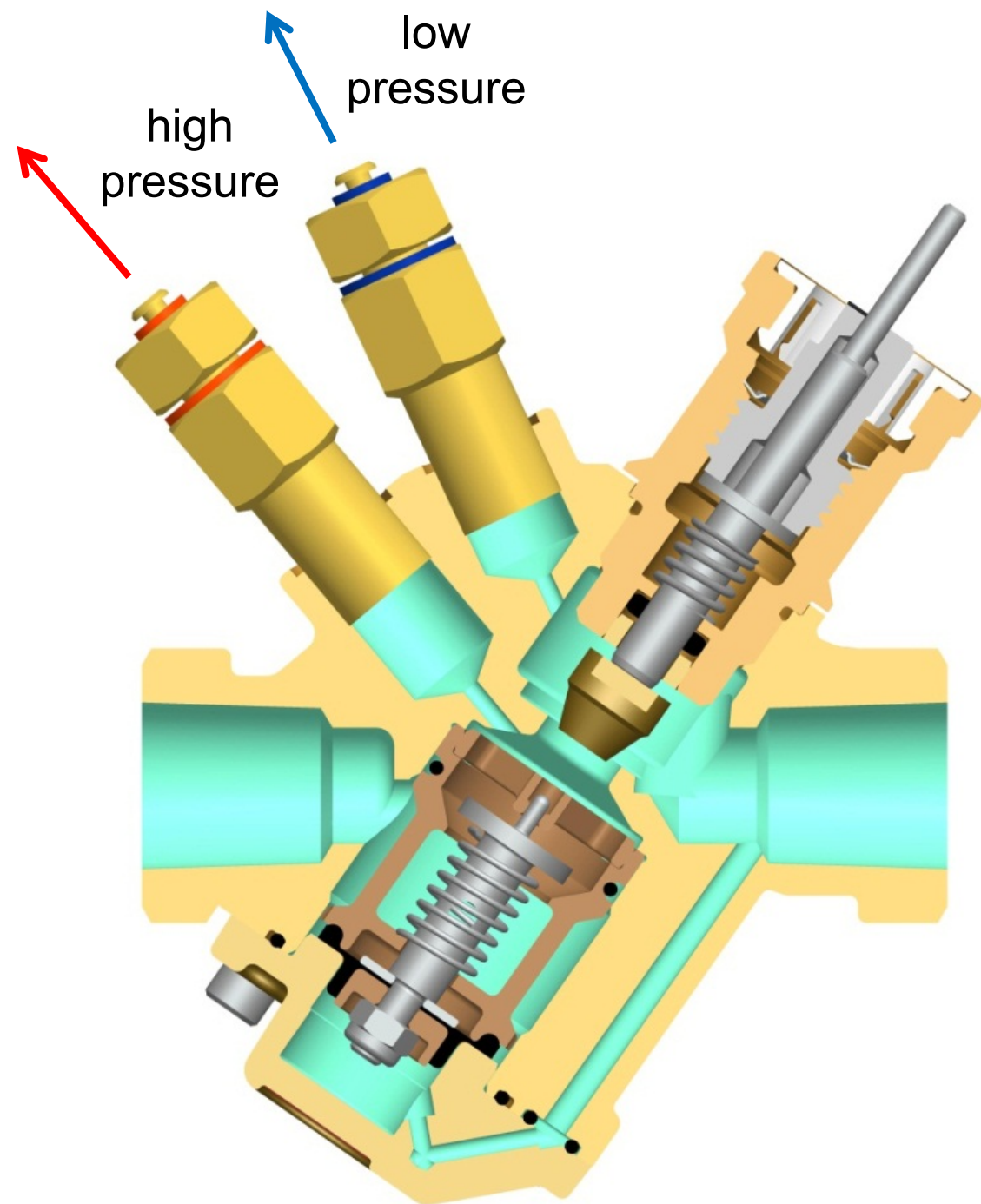
- On / Off
- Modulating

## PICV Commissioning

PICVs are set to give the required flow rate, there is no commissioning required for the PICV

Commissioning Engineers are required to set the pump speed to ensure that the '*least favoured*' (index) PICV generates at least the minimum required differential pressure

if the least favoured PICV has sufficient different pressure, all other PICVs must have greater differential pressure



## Summary

- Change in system design to variable flow controlled by 2 port control valve resulting in pump energy saving
- Fluctuation in system pressure undermines control valve authority
- DPCV installed into sub-circuits to 'protect' control valves from fluctuating pressure to maintain control valve authority
  - terminal units commissioned by conventional proportional method
  - branches commissioned by use of 'Companion' Valve & DPCV
  - branches commissioned independently of each other
- PICVs installed on terminal unit
  - replacing static commissioning valves / 2 port control valve / DPCV